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DAVID W. TAYLOR NAVAL SHIP RESEARCH AND DEVELOPMENT CENTER



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PROPULSION EXPERIMENTS WITH A DEEP TUNNEL PLANING HULL

by

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and

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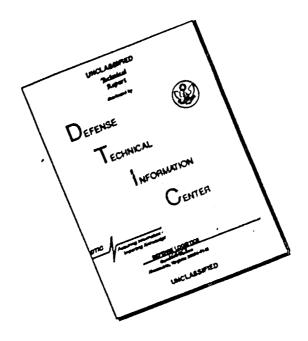
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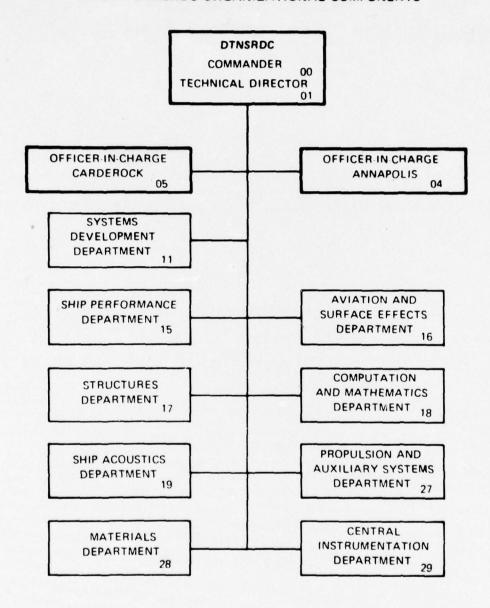
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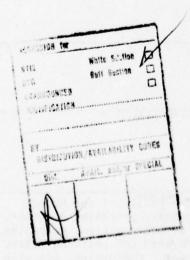
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protection for beaching operations. The 65% tunnel requires the least draft and gives good propeller protection, but it requires more power than the 40% tunnel.



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NOTATION

A _p	Projected area of planing surface, exluding area of external spray strips
Вр	Maximum breadth over chines, exluding external spray strips
D	Propeller diameter
F _{n_V}	Volume Froude number $V/\sqrt{gV^{1/3}}$
g	Acceleration of gravity
J	Advance coefficient, V/nD
K _Q	Torque coefficient
K _T	Thrust coefficient
LCG	Longitudinal center of gravity
LOA	Overall length
L _p	Projected length of chine
n	Propeller rotational speed
P.C.	Propulsive coefficeint, nHxnRxn
PS	Shaft Power
R _T	Total resistance
R _{TAX}	Appendaged hull resistance: $X = P$ for parent hull $X = T$ for tunnel hull
R _{TBX}	Bare hull resistance: X = P for parent hull X = T for tunnel hull
R _{T/} Δ	Resistance coefficient
SHP	Shaft horsepower
t	Thrust deduction fraction $1-(R_T/T)$
T	Thrust
v	Speed

w _Q	Taylor wake fraction determined from torque identity
w _T	Taylor wake fraction determined from thrust identity
▼	Displacement volume
Δ R _T	Displacement weight
$\eta_A \left(= \frac{BP}{R_T} \right)$	Ratio of bare parent hull resistance to resistance with appendages of hull being considered
$\eta_{B} \left(= \frac{BX}{R_{T}} \right)$ $R_{T}AX$	Ratio of bare to appendaged hull resistance for the same hull configuration
$\eta_{C} \left(= \frac{{}^{2}BP}{R_{T}} \right)$ $R_{T}BX$	Ratio of bare parent hull resistance to resistance of bare hull in question
$\eta_{D} \left(= \frac{AP}{R_{T}} \right)$	Ratio of appendaged parent hull resistance to resistance of appendaged hull in question
ⁿ H	Hull efficiency $(1-t)/(1-w_T)$
n _o	Open water propeller efficiency
n _R	Relative rotative efficiency

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ABSTRACT

Resistance and self-propulsion data are presented for Model 5048 fitted with tunnels equal in depth to basic propeller diameter. The relative merits of seven combinations of LCG position, propeller diameter and trim tabs are discussed. A forward LCG position offers the best combination of draft and shaft power. Comparisons are also made with the same hull without tunnels and with two sets of shallow tunnels. The 100 percent tunnel is inferior to both shallower tunnels in draft and power requirements, but it gives superior propeller protection for beaching operations. The 65% tunnel requires the least draft and gives good propeller protection, but it requires more power than the 40% tunnel.

ADMINISTRATIVE INFORMATION

This report was authorized and funded by the Naval Inshore Warfare Craft Office (Code 114) of the Systems Development Department, David W. Taylor Naval Ship R&D Center, which provides Technical Management for the Naval Inshore Warfare Craft Program, SSW-02 (previously the Special Warfare Craft Program, S38-20X). The Principal Development Activity is the Naval Sea Systems Command with program management in the Advanced Technology Systems Division (SEA 03221). Program funding is under element 6.3586 N. This specific task was funded under center work unit 1-1140-606.

UNITS

U.S. customary units were used for the original measurements and calculations. SI (Metric) equivalents of US units are given where they first occur in the text and elsewhere if required for clarity. U.S. units are used alone when their SI equivalents have previously been given and where the unit performs an adjectival function, as in "6 inch propeller". This usage has been adopted to facilitate cross-referencing between this and previous reports in the tunnel-hull series.

The appended data tables, prepared before the adoption of SI units, have not been revised to incorporate SI equivalents due to time and cost constraints.

INTRODUCTION

Minimum navigational draft is a prime requirement for small high performance craft intended for use in shallow water.

Conventional designs have propellers and appendages which project below the baseline, and the usual approach in reducing draft is to reduce propeller diameter or adopt waterjet propulsion, techniques which are not necessarily beneficial in terms of propulsive efficiency.

An alternative approach is to house the propellers and appendages in so-called tunnels, or troughs in the hull bottom. By thus raising the propellers relative to the baseline, the draft can be reduced while retaining the same propeller diameter. Propeller efficiency may even be beneficially affected by the partial shrouding effect of the tunnel wall. On the other hand, the loss of planing surface and changes in pressure distribution on the bottom may adversely affect the running trim, draft and resistance.

The general characteristics of shallow tunnel hull craft were explored by Harbaugh and Blount in an experiment in which Naval Ship Research and Development Center Model 5048 was equipped with two sets of tunnels, accommodated within the original hull lines. Propellers of 6.00 inch (0.152 metre) and 5.25 inch (0.133 metre) diameter were employed. The tunnels were 6.062 inches (0.1540 metres) wide and had depths of 40 and 65 percent of the larger propeller diameter. Resistance and self-propulsion runs were conducted at Langley Field, Virginia.

The current experiment was more specifically intended to examine the potential of deeper tunnels as a means of providing increased propeller protection, especially for beaching operations. For this application overall efficiency is a secondary consideration and waterjets are often employed because of their comparative immunity to damage. It was hoped in this experiment to obtain similar immunity for propellers in deep tunnels while retaining the efficiency advantage of propellers over waterjets.

David W. Taylor Naval Ship R&D Center Model 5048 was fitted with a set of 100 percent deep tunnels and resistance and self-propulsion runs were conducted at Langley Field, for similar conditions of displacement and static trim to the earlier tests. Two other static trim conditions were also investigated, and in addition trim tabs set to 5° and 10° were fitted to the stern for an abbreviated series of runs. The same two sets of propellers used previously were again employed.

This report is in two parts. The first presents the results of the 100 percent tunnel hull experiments while the second compares the results of the current work with the previous parent and tunnel hull test results. Because a large number of hull configurations are discussed, particular care should be taken to distinguish between them. In this report, "parent hull" refers to the original model 5048 without tunnels. Tunnel hull forms are identified by the depth of the tunnels (40,65 or 100%). The 100 percent tunnel hull is further described in accordance with its LCG and trim tab configuration as outlined in Appendix 1. Any of these hull forms is

also described as "bare" or "appendaged" depending on whether the normal appendages (rudder, propeller shafts and struts) were present. Finally, most of the tunnel hull configurations were tested with two sets of propellers.

MODEL DESCRIPTION AND EXPERIMENTAL PROCEDURES

Model 5048 had been selected by Harbaugh and Blount as the parent hull from hydrodynamic and design considerations. The model specifications are given in Table 1. The model was modified to accept two fiberglass tunnels, details of which are shown in Figure 1. The upper curved boundaries were formed by sections of two 6.062 inch (0.1540 metre) diameter cylinders intersecting at 12 degrees, and the side walls were flat vertical sections parallel to the center line. Appendages comprised twin rudders (mounted in the tunnels), propeller shafts and struts.

TABLE 1 - Specifications for Model 5048

LOA	10.125 ft	3.084 m
L _p	9.75 ft	2.972 m
	2.62 ft	0.798 m
B _P XA _P	20.65 ft ²	1.918 m ²
Projected area per rudder	0.078 ft ²	0.00725 m ²
Deadrise (afterbody constant)	8.5 degree	s
Shaft angles (with respect to baseline, for 100% tunnels)	1.17 degre	es

Trim tabs were fitted to the transom for some runs, and details of their location and dimensions are given in Figure 2. Stern views of the hull with 6.00 inch diameter propellers, appendages and trim tabs in place are given in Figure 3.

The propellers were left- and right-handed pairs, having diameters of 6.00 and 5.25 inches (0.152 and 0.133 metres) and nominal tip clearances of 0% and 7.1% of their diameters. Mean values of the open water characteristics of each pair of propellers are given in Figure 4.

The model was ballasted to a displacement of 340.5 lb (154.6 kg) for all runs. It was towed in the thrust line and for propulsion tests it was powered as closely as possible to the self-propulsion point. Five different configurations were tested as described in Appendix 1. Configuration 1 (LCG=39.8% L_p forward of the transom) corresponds to the configuration employed in previous experiments.

EXPERIMENTAL RESULTS FOR 100 PERCENT TUNNEL HULL

The hull characteristics for the five configurations tested are presented in Figures 5 through 9. The data have been non-dimensionalized (except for trim angle) for ease of comparison with previous tests, which were conducted at a slightly higher displacement (345 lb, 157 kg). The draft figures refer to draft of the baseline at station 10. Resistance values have been corrected to fresh water at 59°F (15°C), and only the horizontal component of towing force is reported.

Figure 10 presents the appendage drag factor n_B for configurations 1, 2 and 3. Care should be taken to distinguish between n_B (the ratio of bare to appendaged hull resistance for the same hull form) and n_A (the ratio of the resistance of the bare parent hull to the resistance of the appendaged tunnel or parent hull). Confusion can lead to serious misinterpretation of the data. Addendum 1 discusses this subject at length and should be referred to.

Propulsive characteristics $(1-W_T, 1-W_Q, 1-t, and \eta_R)$ for configurations 1, 2, 3, 4, and 5 are shown in Figures 11 to 14. The computation of these coefficients was done using a computer program available at DTSNRDC which corrected the data to the self-propulsion point. Sample data from the earlier tests were also re-analyzed using this program and identical results to those reported earlier were obtained. The data were cross-faired using another standard DTNSRDC computer program to avoid anomalies arising from visual fairing of individual curves.

Data taken during the experiment are tabulated in Appendix 2.

These data correspond to Langley water conditions, which are as given in the tables. Data which were obviously defective have been omitted, which accounts for missing run numbers.

Propulsive coefficients calculated from the Langley data (but not cross-faired) are presented in Appendix 3.

Trim, draft and propulsive coefficients for the basic condition with 6 inch propellers have not been plotted because their abnormal values and scatter indicate that the measurements are erroneous.

These measurements were the first to be taken during self-propulsion tests and problems with test techniques were encountered.

DISCUSSION OF 100 PERCENT TUNNEL HULL RESULTS

The general trends exhibited in trim, draft and resistance are consistent with the configuration changes represented in Figures 5 through 7. Rearward movement of the LCG results in increased running trim and reduction in the volume Froude number at which peak trim occurs. It also results in greater maximum draft, although the peak remains at $F_{n_{\nabla}} = 1.3$. A secondary hump occurs in the draft curves, roughly coincident with the point of maximum trim for each configuration. Both trim and draft are somewhat higher in the self-propelled condition than when towed. A contrast is evident in the comparisons of bare hull and appendaged hull behaviour, where the bare hull trim and draft are below, equal to or, above the appendaged hull values depending on the static trim. Despite this the appendaged resistance is always higher then the bare hull resistance.

A feature not noted in previous tests is the distinct trim and draft characteristics associated with each set of propellers.

(Previously, one set of curves defined the trim and draft characteristics for both propeller diameters.) This may be due to the increased tunnel area acted on by the propeller flow field.

Comparing appendaged resistances, configuration 1, with the basic LCG position, has the lowest resistance up to $F_{n_{\rm B}}$ =4.0.

Configuration 2, with the LCG moved forward, shows higher resistance over the entire speed range. Configuration 3 shows higher resistance at intermediate speeds but matches configuration 1 from $F_{n_{\nabla}} = 3.0 \text{ to } F_{n_{\nabla}} = 4.0 \text{ and is slightly better at higher speeds.}$

A comparison of Figures 5, 8 and 9 shows that running trim is markedly reduced by the addition of 5° transom flaps. Changing to 10° flaps reduces the peak trim angle still further. Running draft is also reduced, but less dramatically, and at F = 4.0 it is practically unchanged for both flap settings. Resistance characteristics are practically unchanged up to F = 2.5, but above that speed the resistance increases sharply with increasing flap angle.

The appendage drag factor η_B (Figure 10) changes somewhat with LCG position, with configuration 1 giving the highest values, in the order of 0.97. The other configurations yield values in excess of 0.92 over the Froude number range from 2.0 to 4.0 (the accuracy of the calculation becomes questionable at lower Froude numbers because of the small numbers being ratioed). These high values probably reflect the shadowing effect produced by mounting the appendages in deep tunnels out of the free stream flow.

The propulsive characteristics are given in Figures 11 through 14. They exhibit many of the same features that were seen in earlier experiments. For example, n_R is higher for the propeller with zero tip clearance, (1-t) is relatively unaffected by propeller diameter, and (1-W_Q) and (1-W_T) are higher for the smaller diameter propellers. Reference (1) discusses these effects at length, so they will not be elaborated on here.

It should be noted that (1-t) is computed as the ratio of the horizontal component of appendaged resistance to the shaft line thrust, and this is consistent with the earlier tunnel hull data analysis.

Figures 15 and 16 are summary plots for each set of propellers, from which it can be seen that both sets of propellers are quite insensitive to changes in trim and draft arising from shifts in the LCG position, as evidenced by the similarity of the η_R , $(1-W_Q)$ and $(1-W_T)$ curves. The introduction of trim tabs produces greater changes in these coefficients, apparently because the low trim angles which are obtained serve to mask the propellers to greater degree.

PERFORMANCE EVALUATION OF TUNNEL HULL FORMS

The relative merits of the various tunnel hull forms tested during this program are discussed in the following sections. The main text presents comparisons of overall draft and required shaft power, the two most important performance criteria, so that the results of using different tunnels can be readily seen. This is an appropriate method of presentation because all of the models were derived from the same parent hull form.

An alternative method of data presentation is the one adopted in Reference 1. There the objective was to formulate a preliminary design method for obtaining initial powering estimates for tunnel hulls derived from any conventional planing hull form. In this method the bare hull resistance of the basic planing hull in question

is used in conjunction with an efficiency factor $\frac{1}{{}^{\eta}A^{\eta}H^{\eta}R}$ (which accounts for all the interaction effects of the tunnels and propellers with the parent hull) to obtain an estimate of shaft power from

$$P_S = R_{T_{BP}} V(\frac{1}{\eta_A \eta_H \eta_R}) \frac{1}{\eta_O}$$

Values of the efficiency factor were determined from the 40% and 65% tunnel experiments described previously.

Unfortunately, a data reduction error resulted in incorrect values of the efficiency factor being reported in Reference 1.

This also led to misleading conclusions being drawn concerning the advantages of 65% tunnels, as a comparison of Reference 1 and this report will show. The opportunity has been taken in this report to correct these errors and to add data for 100% tunnels so that the usefulness of the design method can be extended. These corrections and supplementary data have been presented in the form of an addendum which can readily be copied and inserted in Reference 1.

Relative Merits of 100 Percent Tunnel Hull Configurations

Two measures of merit can be applied in attempting to evaluate the craft configurations reported. The first is obviously the navigational draft. The value reported here is the draft of the baseline at station 10 (transom), chosen because it represents the deepest point on the hull for the general case of bow-up running trim. Shaft power, obtained from the equation

$$P_S = R_{T_{BX}} V(\frac{1}{\eta_B \eta_H \eta_R}) \frac{1}{\eta_Q}$$

or
$$P_S = R_{T_{AX}} V(\frac{1}{n_H n_R}) \frac{1}{n_O}$$

is the other measure of merit. As the equations make clear, shaft power depends on three separate factors, namely the bare or appendaged resistance of the hull in question, the interaction effects of the propeller and hull, and propeller open-water characteristics. The shaft power calculation in effect summarizes the results of variations in these factors which have already been illustrated individually. Therefore, in assessing the relative merits of the seven combinations of propeller diameters, LCG's and trim tabs under consideration, it is necessary to examine their power requirements to determine the net gains to be achieved by changing any of these parameters.

Power requirements and drafts of the seven configurations are summarized in Figures 17 and 18. Unfortunately, the parent configuration with zero-clearance propellers is not represented because of faulty data, so the comparison is incomplete; nevertheless the trends exhibited in these figures reveal that this configuration can be expected to have slightly lower draft then the same configuration with smaller propellers, and its shaft power input is likely to be about 7 percent lower, for $F_{n_{\rm p}} > 3.0$.

The figures show that the use of 10° trim tabs with zero-clearance propellers gives minimum draft at hump and up to $F_{n_{\nabla}}$ =3.5; however, this also requires the highest P_{S} expenditure at high speed. A better compromise is the use of 5° trim tabs, but the best appears to be the use of a forward LCG position, in which case the draft at hump increases only slightly (although it decreases less rapidly with increasing speed). The latter option requires

significantly less power and avoids mechanical complexity and is thus the most desirable compromise.

Comparison With Previous Tunnel Hull Forms

The same two measures of merit which were employed in the foregoing section can again be used to evaluate the 100% tunnel hull relative to the parent and shallow tunnel hull forms. Figure 19 gives the shaft power requirements for the various hull-propeller combinations, all at LCG = 39.8%. The 100% tunnel - 6 inch propeller power requirements have been estimated from the trends observed during the current series of tests. It can be seen that the power requirements become progressively greater as tunnel depth is increased, to the extent that the 100% tunnel - 5.25 inch propeller combination requires 43% more power than the parent hull without tunnels, at $F_{n_{\rm p}}$ = 4.0.

This increase in required power is due to two factors. First, there is a very marked increase in resistance associated with tunnels greater than 40% of propeller diameter. Figure 20 gives the percentage changes in resistance coefficient for various tunnels compared to the parent hull, both bare and appendaged. The 100% tunnel hull has as much as 50% higher resistance for the bare hull and 32% with appendages. This puts the deep tunnel at a serious disadvantage when computing the shaft power of the various hull forms. The increased resistance can only be offset by substantial increases in propulsive coefficient $(\eta_H x \eta_R x \eta_O)$, and these

unfortunately are not realized in practice (Figure 21). In fact, the opposite is true and this is the second cause of the power increase.

It could be argued that different propellers more suitably matched to the particular operating conditions would improve the relative power requirements of the deep tunnel hull, by achieving a higher η than the propellers actually used in the experiment. (This assumes that $\eta_H x \eta_R$ is not affected by changes in propeller pitch.) However, Figure 21 demonstrates that a sizeable increase in n is required simply to bring the propulsive coefficient back to its original value (i.e., the parent value) before beginning to compensate for the large increase in appendaged resistance. To fully compensate for both effects could require upwards of a 35% improvement in η , and this is unlikely to be achieved in practice. Further experiments with various propellers of the same diameter but with various expanded area ratios and pitches would help in clarifying this point. Meanwhile, it can be accepted that the penalty in shaft power is significant when 100%, and to a lesser degree 65%, tunnels are adapted.

The other merit factor to be considered is navigational draft. Here the comparisons are complicated because the propellers project below the baseline in some cases. Hence, overall draft becomes the criterion rather than draft of the baseline (although the two may coincide). Nevertheless, a comparison of baseline drafts serves as a useful starting point (Figure 22a). The values for the 100% tunnel hull are for the 5.25 inch propellers but the 6.0 inch

propellers would give similar results. It is noteworthy that the baseline draft increases as tunnel depth increases. This is due to the loss of planing area and buoyancy aft and is accompanied by increased trim angles.

A different result is obtained when the overall draft, including propeller draft, is considered (Figure 22b). Here the parent hull clearly is at a disadvantage while the 65% tunnel hull, whose propellers do not project below the baseline, is superior. The 40% tunnels offer a fair advantage over the parent but the 100% tunnels are of minimal benefit. This is clearly due to the loss of buoyancy and planing area of the deep tunnels resulting in much deeper running draft visavis the shallower tunnels. (The estimated displacement of the various tunnels is: 40%, 12 lb; 65%, 17 lb; 100%, 34 lb).

In view of the small improvement in draft over the parent hull and the very large increase in shaft power required, it could be concluded that the 100% tunnel hull is inferior to shallower tunnel hull craft. However, this conclusion can not be considered absolute, for two reasons. First, all the hull-tunnel forms have been compared with LCG held constant and it has been shown, at least for the 100 percent tunnel, that other LCG positions lead to more favourable characteristics. Second, the objective in utilizing 100 percent tunnels was primarily to achieve increased propller protection. The alternative, the use of waterjet propulsion, would also involve a significant performance penalty so that a more

meaningful comparison would be waterjets versus deep tunnel propellers. In the absence of waterjet tests on the same hull, the comparison is not easily made but the propulsive coefficients greater than 0.6 which were recorded for the propellers in deep tunnels are not likely to be matched by waterjets. This is only a preliminary judgement, of course, and should be confirmed by waterjet tests in the same model.

For general operations the 40% and 65% tunnel hulls represent useful compromises of draft reduction versus required shaft power, with the choice depending on the relative importance attached to draft, propeller protection and power requirements. The 65% tunnel hull offers lower draft and greater propeller protection, at the expense of greater shaft power.

A factor to be borne in mind in considering the operation of planing craft in shallow water is the suction effect between the hull and the bottom. Investigation of this phenomenon² shows that sinkage and resistance can be greatly increased at near-hump speeds by decreasing the water depth, but at high speeds the resistance can be significantly reduced. These effects should be remembered when applying the data reported here to actual craft designs. The comparisons made here should, however, remain valid in determining the relative merits of the various hull forms.

CONCLUSIONS

The relative merits of seven combinations of propellers, LCG's and trim tabs have been determined using baseline draft and required shaft power as criteria, for the 100% tunnel hull. Zero-clearance propellers are slightly superior to smaller-diameter propellers. Trim tabs reduce the peak draft, which occurs at about $F_{\eta_{\chi}}$ =1.3, but require greater power at high speed. A forward LCG position gives almost the same peak draft without as great a power demand at higher speeds. An aft LCG position produces the best power characteristic but the greatest draft at hump.

The 100 percent tunnel hull has been compared with 40% and 65% tunnel hulls and with the parent hull without tunnels, using similar criteria. Power requirements become progressively greater as tunnel depth increases, and this is primarily due to increased hull resistance. The 100% tunnel requires about 40% more power than the parent hull at $F_{n_{\nabla}}$ =4.0. Changes in propeller design are unlikely to significantly reduce this margin.

Overall draft (including propellers) is used as another criterion. All the tunnel hulls are superior to the parent. The 100% tunnel is inferior to shallower tunnels but gives greatest protection to the propellers and permits beaching. The 40% and 65% tunnels both offer some advantages and the choice depends on the tradeoffs in draft, power and propeller protection. The 65% tunnel hull offers lower draft and greater propeller protection but requires more power than the 40% tunnel hull.

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One of the authors of this report, Mr. W.E. Ellis, is a Defence Scientist with the Canadian Department of National Defence, on exchange to the David W. Taylor Naval Ship Research and Development Center from the Defence Research Establishment Atlantic during the period September 1975 to August 1977.

APPENDIX 1

Model 5048 with 100 Percent Tunnels

Summary of Test Conditions

Configuration 1 LCG = 3.88 ft (1.18 m) from Station 10 = 39.8% L_p

Bare Hull Resistance

Runs 341-355

Resistance Test (Appendages) Runs 4-24

Powering Test 6.0" Propeller

Runs 26-65

Powering Test 5.25" Propeller Runs 66-83

Configuration 2 LCG = 4.32 ft (1.32 m) from Station 10 = 44.8% L_p

Bare Hull Resistance

Runs 356-377

Resistance Test (Appendages) Runs 84-110

Powering Test 6.0" Propeller Runs 132-150

Powering Test 5.25" Propeller Runs 112-129

Configuration 3 LCG = 3.39 ft (1.03 m) from Station 10 = 34.8% L_p

Bare Hull Resistance

Runs 378-392

Resistance Test (Appendages)

Runs 151-166, 270-274

Powering Test 6.0" Propeller

Runs 168-181

Powering Test 5.25" Propeller

Runs 182-185, 254-269

Configuration 4 LCG = 39.8% Lp

10° Trim Tabs

Resistance Test (Appendages)

Runs 289-305

Powering Test 6.0" Propeller

Runs 276-288

Configuration 5 LCG = 39.8% Lp 5° Trim Tabs

Resistance Test (Appendages)

Runs 326-340

Powering Test 6.0" Propeller

Runs 306-321

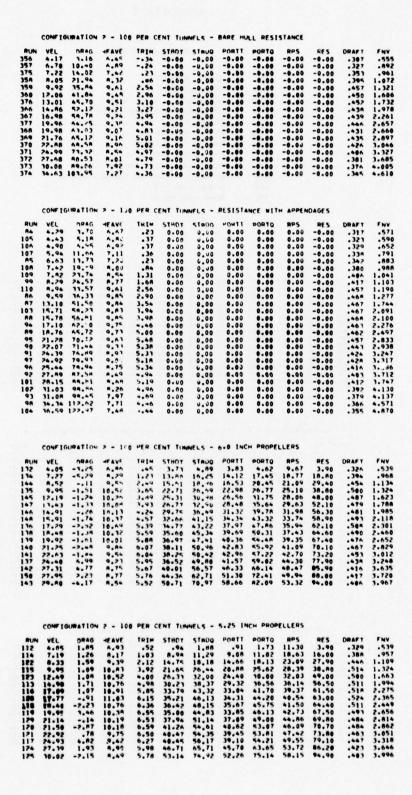
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Appendix 2

EST TO COLUMN EXADENCE

VEL	-	VELOCITY, PT/SEC
PRAG	-	TOWING PORCE, 18.
MAYE	-	BARBLINE DRAFT AT TRANSCH, INCHES
TRDI	-	TRIN ANGLE CHANGE FROM REST, DECREES
STROT	-	THRUST ON STARBOARD STAFT, 15.
STEDQ	-	TORQUE ON STARBOARD START, LB. IN.
PORTT	-	THRUST ON PORT SHAPT, LB.
PORTQ	-	TORQUE ON PORT SHAFT, LB. 19.
	-	SMAPT BOTATIONAL SPEED, REDS/SEC
124	-	APPENDAGED RESISTANCE PRON RESISTANCE TEST, LS.
DRAFT	-	BASELINE DRAFT / V
787	-	TOLARS PRODUCTIONS

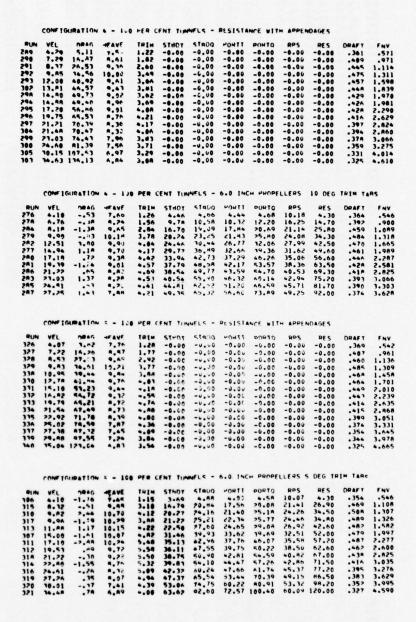
	CONFI	GUP AT I ON	1 - 100	PER C	ENT TUN	NELS -	RARE HU	LL RESI	STANCE			
RUN	VEL	DRAG	FAVE	TRIM	STOOT	STADO	PORTT	PORTO	RPS	RES	DRAFT	FNV
341	4.32	4.91	7.99	1.53	-0.00	-0.00	-0.00	-0.00	-0.00	-0.00	.380	.575
342	7.04	13.61	A. 95	5.50	-0.00	-0.00	-0.00	-0.00	-0.00	-0.00	.425	.937
354	8.76	31.52	10.60	4.02	-0.00	-0.00	-0.00	-0.00	-0.00	-0.00	.504	1.166
343	12.16	41.33	10.67	4.62	-0.00	-0.00	-0.00	-0.00	-0.00	-0.00	.517	1.335
345	14.98	44.56	10.50	5.65	-0.00	-0.00	-0.00	-0.00	-0.00	-0.00	.499	1.994
346	16.88	54.19	10.12	6.09	-0.00	-0.00	-0.00	-0.00	-0.00	-0.00	.481	2.247
347	19.49	40.67	9.61	6.23	-0.00	-0.00	-0.00	-0.00	-0.00	-0.00	.457	2.621
348	21.59	66.98	9.04	0.03	-0.00	-0.00	-0.00	-0.00	-0.00	-0.00	.430	2.874
349	27.99	44.16	A.46	5.77	-0.00	-0.00	-0.00	-0.00	-0.00	-0.00	.421	3.061
350	53.98	67.09	A.42	5.63	-0.00	-0.00	-0.00	-0.00	-0.00	-0.00	.410	3.192
355	24.85	71.40	4.55	5.61	-0.00	-0.00	-0.00	-0.00	-0.00	-0.00	.404	3.308
351	27.26	79.84	7.92	5.10	-0.00	-0.00	-0.00	-0.00	-0.00	-0.00	.374	3.629
353		101.89	7.14	4.24	-0.00	-0.00	-0.00	-0.00	-0.00	-0.00	.350	4.638
					0.00	-0.00			-0.00		,,	41030
		GURATION								-		
RUN	VEL	DRAG	HEAVE	IHIM	STHOT	STAUG	PORTT	PORTO	APS	RES	DRAFT	FNV
15	4.51	4.30	7.07	1.77	0.00	0.00	0.00	0.00	0.00	-0.00	.379	.586
17	7.20	16.99	0.23	2.56	0.00	0.00	0.00	0.00	0.00	-0.00	.439	.600
	7.34	14.52	9.12	2.50	0.00	0.00	0.00	0.00	0.00	-0.00	.431	.977
19	9.77	34.42	10.45	4.61	0.00	0.00	0.00	0.00	0.00	-0.00	.515	1.301
	10.04	14.74	10.70	4.64	0.00	0.00	0.00	0.00	0.00	-0.00	.504	1.336
22	13.19	45.42	10.31	5.23	0.00	0.00	0.00	0.00	0.00	-0.00	.490	1.756
7	13.92	44.72	10.12	5.13	0.00	0.00	0.00	6.06	0.00	-6.66	.461	1.653
	15.64	51.53	10.34	5.83	0.00	0.00	0.00	0.00	0.00	-0.00	.491	5.085
53	16.47	55.25	10.27	4.14	0.00	0.00	0.00	0.00	0.00	-0.00	.484	2.192
24	18.85	50.41	9.77	4.27	0.00	0.00	0.00	0.00	0.00	-0.00	.464	2.509
70	21.79	44.12	0.09	5.8A	0.00	0.00	0.00	0.00	0.00	-0.00	.437	2.901
10	21.49	42.51	4.90	5.99	0.00	0.00	0.00	0.00	0.00	-0.00	.427	2.914
11	24.79	72.73	A. 74	5.50	0.00	0.00	0.00	0.00	0.00	-0.00	. 396	3.300
12	27.71	A1.46	7.81	5.09	0.00	0.00	0.00	0.00	0.00	-0.00	.371	3.689
13	30.42	A9.47	7.71	4.93	0.00	0.00	0.00	0.00	0.00	-0.00	.364	4.049
1:	36.55	115.66	7:27	30	0.00	0.00	•.••	•.••	0.00	-0.00	.345	4.367
	CONF I	GURATION	1 - 100	PFR C	ENT TIM	MELS -	5.25 IN	CH PROP	ELLERS			
RUN	VEL	194G	-EAVE	-	51801	-	PORTT	PORTO	RPS	RES	DRAFT	FNV
72	4.27	.11	4.07	1.69	1.64	5.10	3.37	4.87	11.52	5.10	.301	.560
87	6.90	1.47		2.56	9.24	11.52	9.09	11.34	10.73	15.70	.441	
49	9.41	1.00	10.97	5.13	23.52	26.19	22,45	27.03	25.03	30.00	.521	1.123
01	12.46	1.20	11.29	5,77	27.05	13.43	20.00	31.51	32.00	37.70	.534	1.659
03	10.17	-2.71	10.44	7.37	37.07	47.38	35.70	46.39	41.03	50.30	.507	2.445
74	10.45	5.97	10.41	7.70	34.70	44.24	33.63	43.23	41.09	50.40	.504	2.456
75	19.42	7.72	10.17	7.63	35.61	45.41	34.42	45.80	42.85	61.70	.483	2.630
76	21.11	4.41	4.49	7.10	36.94	50.01	35.69	47.41	44.69	65.10	.470	2.837
77	77.A6	7.40	•,•0	0.96	40.17	53,06	30,49	51.00	47.26	60.90	.447	3.043
73	24.91		A. 90	0.47	41.00	57.62	*0.03	45.41	49.87	73.90	.423	3.316
78	27.38	1.00	7.00	5.01	43.49	40.92	42.13	50.60	52.96	80.50	. 300	3.645
	,	1,-0		3.41	46.04	46.72	*3.07	45.55	50.05	87.10	. 375	3.951



Appendix 2 - (Cont.)

	CONFIG	HEATION	1 - 1 0	-	FNT THE	WAFLS -	-	HL HFS	STANCE			
RUN	VEL	2046	4F AV-	1414	STHOT	STAJO	-0411	PONTO	805	RES	DRAFT	FNV
37A	4.11	17.91	9.07	6.49	-0.00	-0.90	-0.00	-0.10	-0.00	-0.00	.431	.547
100	7.01	15.48	11.85	17	-0.00	-0.00	-0.00	-0.00	-0.00	-0.00	.564	1.166
301	9.97	41.21	11.00	0.49	-0.00	-6.00	-0.00	-0.00	-0.00	-0.00	.564	1.327
JAS	15.91	40.99	11.3'	7.17	-0.00	-0.00	-0.00	-0.00	-0.00	-0.00	.540	1.721
183	17.05	54.55	10.45	7.44	-0.00	-0.00	-0.00	-0.00	-0.00	-0.00	.541	2.013
385	19.41	59.17	9.40	7.31	-0.00	-0.00	-0.00	-0.00	-0.00	-0.00	.450	2.651
386	21.50	47.47	9.14	6.73	-0.00		-0.00	-0.00	-0.00	-0.00	.434	2.474
388	20.95	44.17	4.71	7.44	-0.00	-6.00	-0.00	-0.00	-0.00	-0.00	.367	3.051
389	27.44	71.76	7.41	4.49	-0.00	-0.00	-0.00	-0.00	-0.00	-0.00	. 361	3.674
392	10.00	A0.75	1.74	5.10	-0.00	00	-0.00	-0.00	-0.00	-0.00	.344	4.005
391	14.93	94.72	6.73	4.56	-0.00	-0.00	-3.00	-0.00	-0.00	-0.00	.320	4.650
	CONFIC	UPATION	1 - 1.0	P. R (ENT THE	MIFLS -	4t51514	MICE WIT	H APPF	NDAGES		
RUN	VEL	7846	4FAVE	**	STHOT	Sloug	PO#11	PURTO	PD5	RES	DRAFT	FNV
151	4.46	5.71	9.74	3.10	0.00	v.(0	0.00	0.00	0.00	-0.00	.441	.594
152	7.30	21.43	10.71	4.45	0.00	0.00	0.00	0.00	0.00	-0.00	.509	.972
153	9.78	47. 4	11.85	0.41	0.00	0.00	0.00	0.00	0.00	-0.00	.563	1.302
154	13.23	51.17	11.40	7.32	0.00	0.20	0.00	0.00	0.00	-0.00	.547	1.761
271	14.72	55.13	11.04	7.12	0.00	0.00	0.00	0.00	0.00	-0.00	.524	1.959
155	14.96	41.43	11.01	7.79	0.00	0.00	0.00	2.00	0.00	-0.00	.521	2.159
145	17.46	60.H7	10.20	7.46	0.00	U.30	0.00	0.00	0.00	-0.00	.485	2.324
272	19.74	61.74	7.7-	0.06	0.00	0.00	0.00	0.00	0.00	-0.00	.439	2.581
166	20.68	41.93	9.36	6.09	0.00	0.00	0.00	0.00	0.00	-0.00	.445 .454	2.628
157	22.01	47.17	.0-	6.46	0.00	0.00	0.00	0.00	0.00	-0.00	. 382	2.930
273	24.73	71.80	7.33	5.53	0.00	U.JC	0.00	0.00	0.00	-0.00	.36 7	3.292
159	28.08	A0.62	7.60	5.29	0.00	0.00	0.00	0.00	0.00	-0.00	. 34A	3.310
160	31.06	R9. 16	7.04	4.44	0.00	0.00	0.00	0.00	0.00	-0.00	. 334	4.134
161	36.79	104.67	A.A.	4.17	0.00	0.00	3.00	0.00	0.00	-0.00	.325	4.897
•			•			••••	*****					
RUN 168 170 171 180 169 175 176 177 178 178 174 181	VEL 4.08 7.31 9.85 12.34 14.94 17.04 18.23 19.89 21.30 22.61 24.92 27.12	000 000 000 000 000 000 000 000 000 00	3 - 100 HFAVE 9.42 11.35 12.09 11.95 11.44 10.20 9.53 9.33 #.42 7.46 7.44	PER 0 TRIM 3.35 6.96 7.31 6.10 7.67 7.36 6.27 5.71 5.20	ENT TIM 3.67 15.71 23.12 30.35 31.74 35.84 35.74 35.37 36.17 37.14 40.34 45.12	STRUG 4.48 1d.14 27.44 10.04 17.14 45.32 43.52 47.60 46.72 52.67 54.30 65.37	6.0 INC PONTT 3.85 16.33 23.41 31.36 34.70 34.71 34.92 40.85 42.42 46.54 51.61	PUNDET 5.03 14.24 28,84 43.39 44.58 45.51 60.30 50.77 50.96 64.90 A6.60 76.78	RPS 10.01 20.28 25.50 29.93 33.08 36.06 37.36 40.84 42.74 45.29 48.83 53.05	RES 4.30 20.90 41.80 51.20 56.40 59.60 61.20 63.70 66.10 69.20 73.40 79.30 86.60	DRAFT	FMV .543 .973 1.311 1.643 1.989 2.268 2.427 2.683 3.304 3.637 4.001
	CONF 16	URATION	1 - 150	PER C	ENT TUN	NFLS -	5.25 IN	CH PROP	FLLERS			
RUN	VEL	DRAG	HEAVE	1111	STHOT	STADO	PORTT	PORTO	RPS	or.	00000	
184	3.84	-1.53	9.43	3.35	4.02	5.03	3.80	5.01	11.93	RES	DRAFT	.511
145	14.89	1.22	12.00	8.67	32.7>	40.43	31.69	38.65	44.92	56.40	.570	1.982
163	17.23	-3.61	10.93	6.56	37.35	50.17	37.46	49.64	41.17	72.30	.519	3.241
•					3		30.23	4	40.20	. 2.30	.345	3.241
RUN 256 260 259 261 266 257	VEL 7.29 8.50 10.02 17.44	084G .65 89 44 54	1 - 100 HEAVE 11.44 12.28 12.66 12.10 11.92	TRIM 4.98 6.36 7.17 7.79 8.32 9.56	STROY 15.21 21.84 25.19 28.88 35.22 34.40	\$1900 17.05 25.69 28.74 36.99 43.47	PORTT 15.10 21.86 25.31 29.30 34.60	PORTO 17.35 25.00 29.19 14.34 40.99	RPS 22.70 27.26 30.05 33.68 37.20 37.48	RES 20.80 32.10 42.60 51.50 55.70 56.60	DRAFT .544 .584 .682 .575 .564 .562	FNV .971 1.132 1.334 1.656 1.936
245	15.01	:74	11.47	8.64	36.42	46.78	36.53	43.77	40.25	59.50	.541	2.262
265	19.92	2-10	10.11	7.44	38.92	52.15	40.00	50.38	47.12	63.70	.420	2.652
264	25.00		A. 33	6.45	41.19	57.89	+2.17	53.80	49.86	73.80	. 396	3.328
267	27.42	-1.05	7.50	5.62	47.56	64.56	45.47	58.96	56.83	79.50	.375	3.650
266	15.04	3.61	A. 9A	4.76	56.68	A3.90	58.85	78.62	64.51	102.30	.332	4.665
							1					
							1					
							1					

Appendix 2 - (Cont.)



Appendix 2 - (Cont.)

Appendix 3

			CONF TOUR	T104 1		CENT TH		. DE THE							
MU-MONEL	(E+5)	6.600.FT 1.02100 DIAMFTER		MOI NU-	EL WETTE SHIP (F.	5) = 1.0	E - 16.	920 SQ F	T MODEL	DISPLACE DOEL = 1.	9607	341. LBS	L INE	AR RATIO	• 1.000 .9607
VM(FPS)	EHP	EMP/5HD	SHP	РРМ	JA	KT	JT	1-41	KO	٥ر	1-00	1-7	*	EM	ERR
4.27	.040	.353	-112	646.1	.8412	.3705	.804	.956	-10004	-614	.730	.712	.585	.745	.809
6.98	.199	.417	1-105	1161.2	.7530	.3760	.793	1.008	.08854	.732	.931	.776	.574	.807	.934
12.46	1.015	.484	2.097	1704.5	.7625	.3872	.770	1.010	.08834 .08067	.734 .815	.962	.784	.559	.776	.961
18.37	1.947	.550	3.543	2478.8	1.0163	.2836	.984	.968	.06997	.930	.915	.038	.683	.866	.930
19.82	1.949	.512	3.424	2512.0	1.0446	.2936	1.008	.964	.07087	.921	.921	.778	.672	.807	.935
21.31	2.522	.537	4.695	2755.5	1.0606	.2659	1.021	.963	.06752	1.005	.902	.846	.701	.839	.913
24.91	3,347	.617	6.561	2998.8	1.1392	.2326	1.092	.958	.06048	1.034	.907	.885	.733	.924	.912
29.68	4.700	.653	7.198	3394.8	1.1605	.2069	1.146	.956	.05535	1.058	.909	.915	.754	.957	.910
			CONF I GUHA	TION 2 -	100 PFR	CENT TU	NNELS -	5.25 INC	H PROPEL	LERS					
HODEL LE	NGTH =	6.600 FT			EL WETTE			920 SQ F		DISPLACE		341. LBS		AR RATIO	
NU-MODEL MODEL PR		DIAMETER	. 5.250	IN CO	SHIP (F.	ALLOWAN	CE =0.00	000		FRICTION			#HU-	Smir = 1.	7002
VM(FPS)	EHP	EMP/SHP	SHP	HPM	JA	KT	JT	1-#7	KO	٥ر	1-40	1-7	EP	EM	ERR
4.05	.029	.423	. '65	7-2-1	.7345	.1560	1 - 256	1.701	-04800	1.169	1.583	1.108	.782	.651 .876	.831
7.19 8.33	.209	.477	.663	1148.5	.8585	.3725	.762	.932	.08708	.747	.866	.016	.554	.949	.935
9.95	.702	.573	1.943	1719.4	.7936	.3710	.803	.962	.08632	.755	.951	.945	.580	.982	.947
14.98	1.113	.575	2.675	2148.4	.9364	.3214	.905	.964	.07680	.A56	.912	.920	.640	.955	.941
17.09	2.035	.594	1.253	2374.1	1.0059	.3025	.944	.957	.07316	.996	.907	.904	.662	.945	.939
18.40	2.154	.604	1.567	24nh.A	1.0230	.2870	.977	.955	.07150	.914	.893	.924	.679	.968	.919
19.95	2.44R	.623	4.247	2640.2	1.0524	.2691	1.014	.964	.06727	.960	.912	.930	.698	.965	.925
21.50	2.764	.612	4.517	27 17.0	1.0777	.2633	1.027	.953	.06629	.970	.901	.899	.704	.943	.922
22.92	3.075	.623	5.573	3017.9	1.1018	.248A	1-057	.963	.06403	1.028	.907	.914	.716	.970	.911
27.39	5.180	.63A	A.777	3240.2	1.1593	.2257	1.106	.954	.05995	1.039	-897	.912	.739	.955	.897
			CONE LOUIS	TION 2	- 186 PEP	-	Nur S -	4 . 0 INCH	PROPELL	FPS					
NU-MODEL	L (F+5)	6.600 FT		MOI NU-	- 100 PFR DEL WETTE SHIP (F+	D SURFAC	E - 18.	920 SQ F	T MODEL	DISPLACE	9602	341. LBS		AR RATIO	
NU-MODEL	L (F+5)	6.400 FT	- 6.000	MOI NU-	EL WETTE	D SURFAC	E - 18.	920 SQ F	T MODEL	DISPLACE	9602	341. LBS			
NU-MODEL PR	L (F+5) ROPFLLER EMP	6.600 FT = .98600 DTAMETER EMP/SMP	# 6.000 SHP	IN COL	DEL WETTE SHIP (F* PRELATION JA	D SURFAC 5) = .9 ALLOWAN KT	E = 18. 8600 CE =0.00 JT	920 SQ F 000 1-WT .794	T MODEL RHO-M ITTC KO	DISPLACE ODEL = 1. FRICTION JQ .713	1-W0	1-T	RHO-	SMIP = 1. EM 1.191	.9602 ERR .896
NU-MODEL PE MODEL PE VM (FPS) 4.05 7.27	EHP .029	6.600 FT = .98600 OTAMETER EHP/SHP	# 6.000 SHP	PPM 490.3	DEL WETTE SHIP (F* RRFLATION JA .9911 .8364	D SURFAC 5) = .9 ALLOWAN KT .2520 .2943	E = 18. 8600 CE =0.00 JT .787 .700	920 SQ F 000 1-WT .794 .837	T MODEL RHO-M 11TC KQ .05714	DISPLACE ODEL = 1. FRICTION JO .713 .671	1-W0 .719	1-T	RHO-	EH 1.191 1.030	.9602 ERR .896
NU-MODEL PR MODEL PR VM (FPS) 4.05 7.27 8.52 9.95	EHP .029 .455	6.600 FT = .98690 DTAMETER EMP/SMP .658 .559 .578	= 6.000 SHP .044 .444 .784 1.214	PPM 490.3 1043.0 1203.9 1489.1	DEL WETTE -SHIP (F- PRELATION JA .9911 .8364 .8889 .8024	D SURFAC 5) = .9 ALLOWAN KT .2520 .2943 .2945 .2918	E = 18. 8600 CE =0.00 JT .787 .700 .700 .706	920 SQ F 000 1-WT .794 .837 .865 .879	T MODEL RHO-M ITTC KO .05714 .06044 .06021 .05703	DISPLACE ODEL = 1. FRICTION JQ .713 .671 .674 .714	1-W0 1-W0 .719 .803 .834	1-T .946 .863 .918	EP .617 .564 .564 .568	EM 1.191 1.030 1.061 1.003	.9602 ERR .896 .962 .966
NU-MODEL PR VM (FPS) 4.05 7.27 8.52 9.95 12.19	EHP .029 .455 .702 1.064	6.400 FT = .08600 DIAMETER EMP/SMP .458 .559 .578 .576 .455 .678	SHP .044 .144 .784 1.718 1.62*	PPM 490.3 10~3.0 12°3.9 1488.1 1671.0	DEL WETTE -SMIP (F+ RRFLATION JA .9911 .8364 .8089 .8024 .8754	D SURFAC 5) = .9 ALLOWAN KT .2520 .2943 .2945 .2918	E = 1A. 8600 CE =0.00 JT .787 .700 .700	920 SQ F 0000 1-WT .794 .837 .865	T MODEL RHO-M ITTC KO .05714 .06044	DISPLACE OOEL = 1. FRICTION JQ .713 .671 .674 .714	1-W0 1-W0 .719 .803	1-T .946 .863 .918 .882	EP .617 .564 .564 .568 .600	EH 1.191 1.030 1.061 1.003 1.096	.9602 ERR .896 .962 .966 1.012
NU-MODEL PR VM (FPS) 4.05 7.27 8.52 9.95 12.19 13.43 14.91	EHP .029 .249 .455 .702 1.064	6.400 FT = .98400 DTAMETER EMP/SMP .559 .578 .576 .455 .455 .456	5HP .044 .144 .778 1.918 1.62* 1.676 2.364	PPM 490.3 1043.0 1203.9 1488.1 1671.0 1705.0 1916.3	DEL WETTE -SHIP (F+ RRFLATION JA .9911 .8364 .8089 .8024 .8754 .9131 .9337	D SURFAC 5) = .9 ALLOWAN KT .2520 .2943 .2945 .2918 .2661 .2541 .2541	E = 1A. 8600 CE =0.00 JT .787 .700 .700 .706 .758 .763 .805	920 SQ F 000 1-WT .794 .837 .865 .879 .866	MODEL RHO-M ITTC KQ .05714 .06044 .06021 .05703 .05373 .05266 .05065	DISPLACE ODEL = 1. FRICTION JQ .713 .671 .674 .715 .769 .794	.9602 USED 1-WQ .719 .803 .834 .890 .863 .842 .851	1-T .946 .863 .918 .882 .949 .967	RHO- EP .617 .564 .568 .600 .614 .627	EM 1.191 1.030 1.001 1.003 1.006 1.120 1.074	.9602 ERR .896 .962 .966 1.012 .996 .979
NU-MODEL MODEL PR VM (FPS) 4.05 7.27 8.52 9.95 12.19 13.43 14.91 15.91	EHP .029 .455 .702 1.064 1.272 1.526 1.704	6.400 FT = .98600 DIAMETER EMP/SMP .558 .559 .578 .655 .678 .661	5MP .044 .144 .178 1.218 1.62* 1.676 2.304 2.622 2.622	HOI NU: IN CO PPM 440.3 1043.0 1203.9 1488.1 1671.0 1705.0	DEL WETTE -SHIP (F*- RRELATION JA .9911 .8364 .8089 .8024 .8754 .9131 .9337	D SURFAC 5) = .9 ALLOWAN KT .2520 .2943 .2945 .2918 .2661 .2541	E = 1A. 8600 CE =0.00 JT .787 .700 .700 .706 .758 .783	920 SQ F 0000 1-WT .794 .837 .865 .879 .866	T MODEL RHO-M ITIC KQ .05714 .06044 .06021 .05703 .05373 .05266	DISPLACE ODEL = 1. FRICTION JQ .713 .671 .674 .714 .755	.9602 USED 1-WQ .719 .803 .834 .890 .863	1-T .946 .863 .918 .882 .949	EP .617 .564 .568 .600 .614 .627 .634	EM 1.191 1.030 1.061 1.003 1.096 1.128	.9602 ERR .896 .962 .966 1.012 .996
NU-MODEL PR VM (FPS) 4.05 7.27 8.52 9.95 12.19 13.43 14.91 15.91 17.29 18.48	EHP .029 .249 .455 .702 1.064 1.272 1.526 1.704 1.952	6.48C FT	5HP .044 .744 .744 1.214 1.624 1.676 2.622 2.622 3.475	PPM 490.3 10~3.0 1263.9 1469.1 1671.0 1765.0 1916.3 2008.4 2134.6 2733.7	DEL WETTE -SMIP (F* RRELATION JA .9911 .8364 .8089 .8024 .8754 .9131 .9337 .9906 .9723	D SURFAC 5) = .0 ALLOWAN KT .2520 .2943 .2945 .2918 .2661 .2541 .2432 .2370 .2255 .2171	E = 18. 8600 CE =0.00 JT .787 .700 .706 .758 .805 .818 .842 .860	920 SQ F 0000 1-WT .794 .837 .865 .879 .866 .858 .861 .866	MODEL RHO-M 1TTC KQ .05714 .06044 .06021 .05703 .05373 .05266 .05065 .0496 .04725 .04607	DISPLACE ODEL = 1. FRICTION JQ -713 -671 -674 -714 -755 -769 -794 -803 -851	.9602 USED 1-WQ .719 .803 .834 .890 .863 .842 .851 .844 .866 .857	1-T .946 .863 .918 .882 .949 .967 .926 .905	RHO- EP .617 .564 .568 .600 .614 .627 .634	EM 1.191 1.030 1.061 1.093 1.096 1.120 1.074 1.051 1.026 1.012	.9602 ERR .896 .962 .966 1.012 .996 .979 .982 .975 .991
NU-MODEL PR WM (FPS) 4.05 7.27 8.52 9.95 12.19 13.43 14.91 15.91 17.29	EHP .029 .455 .702 1.064 1.272 1.526 1.704 1.952 2.171 2.441	6.48C FT	5HP .044 .744 .744 1.214 1.624 1.674 2.622 2.622 3.737 3.716	MOI NU. IN COI PPM 490.3 10-3.0 12-3.9 1489.1 1671.0 1916.3 2006.4 2233.7 2347.5	DEL WETTE -SMIP (F+ RRELATION JA .9911 .8364 .8089 .8024 .8754 .9131 .9337 .9906 .9729 .9028 1.0103	D SURFACE 5) = .9 ALLOWAN KT .2520 .2943 .2945 .2661 .2541 .2432 .2370 .2255 .2171 .2014 .1921	E = 1A. 8600 ICE =0.00 JT .787 .700 .700 .758 .860 .818 .860 .892 .912	920 SQ F 0000 1-WT .794 .837 .865 .879 .866 .858 .863 .861 .866 .866	MODEL RHO-M ITTC KQ .05714 .06044 .06021 .05703 .05373 .05266 .05065 .0496 .04725 .04607 .04435 .04246	DISPLACE ODEL = 1. FRICTION JQ .713 .671 .674 .755 .769 .794 .803 .836 .851 .873 .896	.9602 USED 1-WQ .719 .803 .834 .890 .863 .842 .851 .860 .857 .857	1-T .946 .863 .918 .882 .949 .967 .926 .905 .808 .876 .897	RHO- EP .617 .564 .568 .608 .614 .627 .634 .646 .659	EM 1.191 1.030 1.061 1.003 1.006 1.120 1.074 1.051 1.016 1.016 1.016 1.016	ERR .896 .962 .966 1.012 .966 1.012 .979 .982 .975 .991 .985
NU-MODEL PR VM (FPS) 4.05 7.27 8.52 9.95 12.19 13.43 14.91 15.91 17.29 18.48 19.92 21.25 22.63	EHP .029 .455 .702 1.064 1.272 1.704 1.952 2.171 2.441 2.708	6.400 FT = .08600 PT = .08600 PT AME TER EMP/SMP .658 .578 .578 .650 .650 .650 .657 .657 .707	5HP .044 .044 .044 .041 .078 1.214 1.624 1.676 2.622 2.622 2.675 3.776 4.724 4.755	MOI NU 10 10 10 10 10 10 10 10 10 10 10 10 10	DEL WETTE -SHIP (F* RRELATION JA .9911 .8364 .8089 .8024 .8754 .9737 .9986 .9729 .9181	D SURFACE 5) = .9 ALLOWAN KT .2520 .2943 .2945 .2919 .2661 .2541 .2432 .2370 .2255 .2371 .2014 .1921 .1793	E = 18.8 8600 CE =0.00 JT .767 .700 .700 .763 .805 .818 .842 .862 .912	920 SQ F 0000 1-WT .794 .837 .865 .879 .866 .853 .861 .866 .868 .863	T MODEL RHO-M ITTC KQ .05714 .06044 .06021 .05703 .05373 .05266 .04996 .04726 .04435 .04246 .03474	DISPLACE ODEL = 1. FRICTION JQ .713 .671 .674 .714 .755 .769 .794 .803 .836 .851 .873 .496	.9602 USED 1-WQ .719 .803 .834 .890 .863 .842 .851 .844 .868 .857 .857	1-T .946 .863 .918 .882 .949 .967 .926 .888 .876 .892	RHO- EP .617 .564 .568 .600 .614 .627 .634 .654 .654	EM 1.191 1.030 1.061 1.093 1.096 1.128 1.074 1.051 1.026 1.012 1.018	.9602 ERR .896 .962 .966 1.912 .996 .979 .982 .975 .991 .985 .964
NU-MODEL PR VM (FPS) 4.05 7.27 8.52 9.95 12.19 13.43 14.91 17.29 18.48 19.92 21.25 22.40 27.31	EMP .029 .249 .249 .502 1.064 1.272 1.704 1.774 1.774 1.704 1.704 1.704 1.704 1.704 1.704	6.660 FT08600 PT AME TER FMP/SMP .559 .579 .576 .559 .650 .650 .657 .657 .657 .657 .657 .657 .657 .657	= 6.000 SHP .044 .745 1.024 1.624 2.622 2.475 3.127 3.716 4.754 4.256	MOI NU IN COI	DEL WETTE SHIP (F* RRELATION JA .9911 .8364 .8084 .8084 .9131 .9337 .9386 .9723 .9426 .9723 .9426 .9721 .9426 .9721 .9426 .9721 .9426 .9721 .9426 .9721 .9426 .9721 .9426 .9721 .9426 .9721 .9426 .9721 .9426 .9721 .9426	D SURFAC 5) = .9 ALLOWAN KT .2520 .2945 .2915 .2641 .2541 .2542 .2370 .2255 .2171 .2014 .1793 .1714	E = 18.88600 CE =0.00 JT .767 .700 .706 .758 .818 .842 .860 .862 .912 .938 .955 .989	920 SQ F 0000 1-WT .794 .837 .865 .879 .866 .886 .861 .866 .876 .876 .876 .876 .876	TT MODEL RHO-M ITTC KO .05714 .06021 .05703 .05266 .0565 .0496 .04725 .0496 .04725 .0496 .03974 .03922 .03642 .03922	DISPLACE ODEL = 1. FRICTION JQ .713 .671 .674 .714 .755 .769 .794 .803 .836 .873 .896 .970	9002 USED 1-W0 .719 .803 .834 .890 .863 .842 .851 .844 .857 .857 .859	1-T .946 .863 .918 .882 .949 .967 .926 .905 .888 .872 .892 .892	RHO- EP .617 .564 .568 .600 .614 .627 .634 .646 .659 .677 .686 .691 .700	EM 1.191 1.030 1.061 1.003 1.066 1.126 1.074 1.051 1.026 1.012 1.018 1.026 1.027 1.059	.9002 ERR .896 .962 .966 1.012 .996 .979 .902 .975 .905 .964 .970 .982 .961
NU-MODEL PR VM (FPS) 4.05 7.27 8.52 9.95 12.19 13.43 14.91 15.91 17.29 18.48 19.92 21.25 22.63 24.40	EMP .029 .249 .249 .570 1.272 1.004 1.272 1.704 1.272 1.704 1.724 1.952 2.171 2.441 2.708 3.012	6.600 FT08600 PT AWETER FMP/SMP .559 .578 .578 .678 .666 .650 .666 .657 .678 .662 .710 .662 .710 .662 .710 .678	= 6.000 SHP .044 .744 1.214 1.624 1.674 2.629 2.625 3.127 3.124 4.256	MOI NU. IN COI PPM 490.3 10-3.0 1203.9 1408.1 1671.0 1916.3 2018.6 2134.6 2233.7 2347.5 2445.4 2547.9	DEL WETTE SHIP (F* RRELATION JA .9911 .8364 .8084 .8024 .8754 .9131 .9337 .9928 .9729 .9928 1.0658 1.0658	D SURFACE 5) = .9 ALLOWAN KT .2520 .2943 .2945 .2919 .2661 .2432 .2370 .2171 .2114 .2171 .2171 .2171 .2171 .2171	E = 18.88600 CE =0.00 JT .787 .700 .700 .706 .788 .788 .805 .818 .860 .892 .912 .938 .955	920 SQ F 0000 1-WT .794 .865 .879 .866 .863 .861 .866 .863 .861 .866 .874 .841	TT MODEL RHO-M 11TC K0 .05714 .06021 .05703 .05373 .05266 .04725 .04407 .04435 .04246 .0372	DISPLACE ODEL = 1. FRICTION JO .713 .671 .674 .714 .755 .769 .836 .851 .873 .896 .936	.9602 USED 1-WQ .719 .803 .834 .809 .803 .842 .851 .844 .860 .857 .857	1-T .946 .863 .918 .882 .949 .967 .926 .905 .888 .876 .897 .926	RHO- EP .617 .564 .568 .600 .614 .627 .634 .654 .669 .677 .686	EH 1.191 1.030 1.061 1.093 1.096 1.128 1.074 1.051 1.026 1.018 1.026 1.027	.9002 ERR .896 .962 .966 1.012 .979 .982 .979 .981 .991 .984 .970 .982
NU-MODEL PR VM (FPS) 4.05 7.27 8.52 9.95 12.19 13.43 14.91 15.91 17.29 18.48 19.92 21.25 22.63 24.40 27.31 27.95	EHP .029 .249 .455 .702 1.064 1.272 1.576 1.272 1.576 1.272 1.576 1.272 1.576 1.272 1.576 1.272 1.576 1.272 1.576 1.272 1.576 4.275 4.275	6.600 FT08600 PT AWETER FMP/SMP .559 .578 .578 .678 .666 .650 .666 .657 .678 .662 .710 .662 .710 .662 .710 .678	- 6-000 SHP .064 .784 1.214 1.624 1.674 2.622 2.672 2.673 3.327 3.716 4.756 4.756 6.594 6.594 6.594 7.430	MOINT NO COINT NO COI	DEL WETTE- SHIP (F* RRELATION JA .911 .8364 .8089 .8024 .8754 .9131 .9337 .9986 .9723 .9428 1.8658 1.8658 1.1135	D SURFACE D SURFACE - 94 ALLOWAN KT - 2520 - 2945 - 2916 - 2541 - 2432 - 2370 - 2255 - 2171 - 2014 - 1921 - 1793 - 1714 - 1559 - 1529	E = 18.8600 JJ .767 .760 .706 .788 .783 .805 .818 .842 .860 .892 .912 .938 .941 .944	920 SQ F 0000 1-WT .794 .837 .865 .879 .866 .866 .866 .866 .876 .874 .881 .882	T MODEL RHO-MITTC KQ	DISPLACE ODEL = 1. FRICTION JQ .713 .671 .674 .714 .755 .769 .794 .803 .836 .873 .873 .873 .873 .873 .873 .873 .873	9602 USED 1-WC -719 -803 -834 -869 -863 -857 -857 -857 -857 -857 -857 -857 -857	1-T .946 .863 .918 .882 .949 .967 .926 .888 .876 .897 .924	RHO- EP .617 .564 .564 .668 .614 .627 .634 .654 .669 .677 .686 .691 .708	EM 1.101 1.030 1.000 1.128 1.074 1.051 1.026 1.026 1.018 1.026 1.027 1.027 1.018	.9602 EBR .896 .966 .966 .966 .979 .982 .975 .991 .984 .976 .976 .976 .976 .976 .976 .976 .976
NU-MODEL PR MODEL PR VM (FPS) 4.05 7.27 8.52 9.95 12.19 13.43 14.91 15.91 17.29 18.48 19.92 21.25 22.63 24.40 27.31 27.95 29.86	ENGTH = L (F*5)	6.600 FT08600 PT AWETER FMP/SMP .559 .578 .578 .678 .666 .650 .666 .657 .678 .662 .710 .662 .710 .662 .710 .678	- 6-000 SHP - 064 - 788 1-218 1-676 1-676 2-672	MOINT	DEL WETTE (F-SHIP (F-S	D SURFACE 5) = .9 ALLOWAN KT .2520 .2943 .2945 .2919 .2661 .2541 .2541 .2541 .2571 .2014 .1513 .1714 .1553 .1714 .1553 .1589 .1529	E = 1A. 8600 JT .787 .700 .700 .706 .788 .783 .805 .818 .822 .842 .912 .938 .941 .994	920 SQ F 1-WT .837 .865 .879 .866 .853 .861 .866 .876 .8	T MODEL RHO-MITTC KQ .05714 .06044 .06021 .05703 .05373 .05266 .05065 .04996 .04435 .04496 .03972 .03642 .03726 .03758	DISPLACE ODEL = 1. FRICTION JQ .713 .671 .674 .714 .755 .769 .794 .803 .836 .873 .873 .873 .873 .873 .873 .873 .873	9002 USED 1-W0 -719 -803 -804 -806 -807 -807 -857 -857 -872 -866	1-T -946 -863 -918 -882 -947 -926 -888 -876 -897 -924 -940 -940 -940 -940	RHO- EP .617 .564 .568 .601 .614 .627 .634 .654 .667 .666 .671 .686 .701	EM 1.101 1.030 1.000 1.128 1.074 1.051 1.026 1.026 1.018 1.026 1.027 1.027 1.018	.9602 ERR .896 .962 .966 .962 .966 .976 .976 .976 .975 .975 .961 .976 .976 .976 .976 .976 .976 .976 .976
NU-MODEL PR MODEL PR VM (FPS) 4.05 7.27 8.52 9.95 12.19 13.43 14.91 15.91 17.29 18.48 19.92 21.25 22.63 24.40 27.31 27.95 29.86	ENGTH = L (F*5)	6.600 FT98600 FT98600 FT98600 FT98600 FT9800 F	- 6-000 SHP - 064 - 784 1-714 1-62- 1-67- 1-716 2-309 2-672 2-57 3-176 4-756 5-309 6-309 6-309 6-309 7-430	MOINT	DEL WETTE -SHIP (F* -SRELATION	D SURFACE 5) = .9 ALLOWAN KT .2520 .2943 .2945 .2919 .2661 .2541 .2541 .2541 .2571 .2014 .1513 .1714 .1553 .1714 .1553 .1589 .1529	E = 1A. 8600 JT .787 .700 .700 .706 .788 .783 .805 .818 .822 .842 .912 .938 .941 .994	920 SQ F 1-WT .837 .865 .879 .866 .853 .861 .866 .876 .8	T MODEL RHO-M ITTC K0	DISPLACE ODEL = 1. FRICTION JQ .713 .671 .674 .714 .755 .769 .794 .803 .836 .873 .896 .936 .936 .936 .936 .936 .936 .936 .9	9002 USED 1-W0 -719 -803 -804 -806 -807 -807 -857 -857 -872 -866	1-T -946 -863 -918 -882 -947 -926 -888 -876 -897 -924 -940 -940 -940 -940	RHO- EP .617 .564 .568 .601 .614 .627 .634 .654 .667 .666 .671 .686 .701	SMIP = 1. EM 1.191 1.030 1.081 1.081 1.092 1.128 1.074 1.051 1.022 1.018 1.020 1.018 1.059 1.018	.9602 ERR .896 .962 .966 .962 .966 .976 .976 .976 .975 .975 .961 .976 .976 .976 .976 .976 .976 .976 .976
NU-MODEL PR VM (FPS) 4.05 7.27 8.52 9.95 12.19 13.43 14.91 15.91 17.29 18.48 19.92 21.49 27.31 27.95 29.80 MODEL LL NU-MODEL MODEL PR VM (FPS)	ENGTH = L (F-5) FMP032	6.600 FT = .08600 PT AMETER FMP/SMP .559 .576 .659 .677 .661 .650 .657 .677 .667 .707 .665 .710 .678 .710 .685	= 6.000 SHP .064 .78F 1.21F 1.62F 1.67F 2.52C 2.527 3.727 3.716 6.756 4.756 4.756 4.756 7.430	MOI NU. IN COI PPM 1043.3 1043.0 1263.9 1468.1 1671.0 1765.0 1916.3 2006.6 2130.6 2239.7 230.7 240.7 2	DEL WETTE (F- SHIP (F- SHIP (F- SRELATION JA 9911 8364 8089 8024 8754 9131 9137 9189 1.0458 1.0754 1.1135 1.1274 - 106 PFH DEL WETTE SHIP (F- WPFLATION JA	D SURFACES 3 = .9 ALLOWAN KT .2520 .2943 .2641 .2521 .25	E = 18.8600 CE =0.00 JT .767.700 .700 .706 .788 .789 .805 .818 .842 .8600 .802 .912 .938 .945 .940 .941 .940 .940 .940 .940 .940 .940 .940 .940	920 SQ F 0000 1-WT .794 .837 .865 .879 .866 .853 .861 .860 .876 .8	TT MODEL RHO-M ITTC KQ	DISPLACE OOEL = 1. FRICTION JO .713 .671 .674 .714 .755 .769 .794 .803 .836 .851 .873 .896 .970 .900 .976 EPS .DISPLACE OOEL = 1. FRICTION	9002 USED 1-W0 -719 -803 -804 -809 -803 -804 -805 -805 -805 -806 -807 -807 -807 -806 -806 -806 -807 -806 -806 -806 -806 -806 -806 -806 -806	1-T .946 .863 .918 .982 .967 .926 .905 .888 .876 .892 .897 .724 .908 .907 .908	RHO- EP .617.564 .568 .568 .600 .614 .627 .636 .669 .770 .676 .691 .700 .EINERO- EP .575	SMIP = 1. EM 1.191 1.030 1.061 1.083 1.096 1.128 1.071 1.081 1.081 1.09	.9602 ERR .896 .902 .906 1.012 .906 .975 .907 .905 .904 .908 .909 .909 .909 .909 .909 .909 .909
NU-MODEL PR VM (FPS) 4.05 7.27 8.52 9.95 12.19 13.43 14.91 15.91 17.29 18.48 19.92 21.25 22.63 27.31 27.95 29.80 MODEL LI NU-MODEL MODEL PR VM (FPS) 4.08 7.31 9.85	(F-5) ROPFLLER	6.600 FT = .98600 PT AME TER FWP/Sup .559 .576 .655 .676 .655 .677 .661 .650 .666 .652 .710 .678 .710 .710 .710 .710 .710 .710 .710 .710	= 6.000 SHP .064 .78F 1.21F 1.62F 1.67F 2.622 2.622 2.622 2.623 3.727 3.72F 4.256 6.268 6.268 7.430 6.268 7.430 6.268 6.268 7.430	MOI NU. IN COI PPM 10-3.0 12-3.9 14-89.1 1671.0 1716.3 20:68.6 213-6 213-6 213-6 213-6 213-6 213-7 30:12-0 3171.0 4100.3 MO NU IN CO 429M 569.7 11-0.6 151-2.1	DEL WETTE -SHIP (F- SRELATION JA 9911 -8364 -8089 -8754 -9131 -9337 -9763 -91658 1.0658 1.1154 -1115 -110 PFH DEL WETTE -SHIP (F- WPFLATION JA ,RSV4 -7507 -7707	D SURFACES 3 - 9 ALLOWAN KT .2520 .2943 .2943 .2943 .2941 .254	E = 18. 8600 CE =0.00 JT .767 .700 .700 .706 .758 .865 .818 .842 .860 .802 .912 .918 .945 .941 .945 .941 .945 .941 .945 .945 .941 .945 .941 .945 .941 .945 .941 .945 .941 .945 .941 .945 .941 .945 .941 .945 .941 .945 .941 .945 .941 .945 .941 .945 .941 .945 .946 .947 .947 .947 .947 .948 .948 .948 .948 .948 .948 .948 .948	920 SQ F 0000 1-WT .794 .837 .865 .879 .866 .853 .861 .866 .876 .8	TT MODEL RHO-M ITTC KO	DISPLACE ODEL = 1. FRICTION JQ .713 .671 .674 .714 .755 .769 .794 .803 .836 .873 .896 .936 .936 .936 .936 .936 .936 .936 .9	9002 USED 1-W0 -719 -803 -804 -806 -807 -807 -807 -807 -807 -807 -807 -807	1-T .946 .863 .918 .882 .997 .925 .888 .876 .897 .924 .997 .928	RHO- EP .617.564 .568.568 .627.636 .654.666 .656.701 .6701 .6701 .6701 .6701 .6701 .6701 .6701 .6701	SMIP = 1. EH 1.191 1.039 1.091 1.093 1.095 1.076 1.126 1.071 1.051 1.051 1.051 1.051 1.051 1.058 1.018 1.018 AR RATIO SHIP = 1. EH .015	.9602 ERR .896 .902 .906 1.012 .906 .979 .905 .907 .905 .908 .908 .909 .909 .909 .909 .909 .909
NU-MODEL PR WM (FPS) 4.05 7.27 8.52 9.95 12.19 13.43 14.91 17.29 18.47 19.92 21.25 22.63 27.31 27.95 29.80 MODEL LI NU-MODEL MODEL PR WM (FPS) 4.08 7.31 9.85	(F-5) ROPFLLER	6.600 FT = .98600 PT AME TER FWP/Sup .559 .576 .655 .676 .655 .677 .661 .650 .666 .652 .710 .678 .710 .710 .710 .710 .710 .710 .710 .710	= 6-000 SHP .064 .764 .776 .121 .121 .127 .127 .127 .127 .127 .127	MOIN 001 NO. 1 NO. 2 NO.	DEL WETTE -SHIP (FSHIP	D SURFACE 5) = .9 ALLOWAN KT .2520 .2943 .2945 .2943 .2945 .2918 .2651 .2918 .2651 .2918 .2651 .2918	E = 1A.8600 JT .767 .700 .700 .700 .700 .706 .758 .865 .862 .862 .955 .989 .981 .994 JNNELS	920 SQ F 1-WT .794 .865 .879 .866 .858 .861 .866 .866 .866 .866 .866 .866 .86	T MODEL RHO-M ITTC KO .06154 .03793 .03726 .03789 .	DISPLACE OOFL = 1. FRICTION JQ .713 .671 .674 .714 .755 .769 .704 .803 .836 .873 .876 .936 .936 .937 .936 .976 EPS	9002 USED 1-W0 -719 .003 .034 .003 .034 .003 .003 .004 .003 .005 .005 .005 .005 .005 .005 .005 .005	1-T .946 .863 .918 .882 .947 .926 .905 .878 .878 .897 .924 .947 .940 .940 .940 .947 .940 .947 .940 .947 .941 .941	RHO- EP .617.564.568.6509 .614.627.634.666.6509 .677.1686.6791 .701 LIMERHO- EP .575.556.569	SMIP = 1. EH 1.191 1.039 1.061 1.093 1.095 1.076 1.126 1.071 1.051 1.051 1.051 1.051 1.051 1.058 1.018 1.018 AR RATIO SMIP = 1. EH .015 .016 .016 .016 .016 .016 .017	PR
NU-MODEL PS WM (FPS) 4.05 7.27 8.52 9.95 12.19 13.43 14.91 15.91 17.29 21.25 22.63 27.31 27.95 29.80 MODEL LI NU-MODEL MODEL PS WM (FPS) 4.08 7.31 9.85 12.34 14.94 17.04	(F+5) ROPFILER	6.600 FT = .98600 FT AME TER FMP/SMP .550 .576 .550 .676 .650 .660 .652 .657 .707 .662 .657 .673 .707 .685 .665 .675 .675 .675 .675 .675 .675 .67	= 6.000 SHP .044 .746 .716 .1216 .1216 .122 .475 .3127 3.127 3.127 3.127 3.127 3.127 3.127 3.127 3.127 3.126 4.256 6.294 6.294 6.294 6.296 7.430 COMFIGURE SHP .400 .500	MOIN 001 NO. 1 NO. 2 NO.	DEL WETTE -SHIP (FSHIP (FSHIP (FSHIP) (FSHIP (FSHIP) (FSHIP (FSHIP)	D SURFACE 5) = .9 ALLOWAN KT .2520 .2943 .2945 .2943 .2945 .2919 .2651 .2919 .2651 .2919 .2171	E = 1A.8600 JT .767 .700 .700 .700 .700 .706 .758 .805 .805 .805 .805 .805 .905 .906 .906 .906 .906 .906 .906 .906 .906	920 SQ F 1-WT .794 .865 .879 .866 .858 .861 .866 .866 .866 .866 .866 .866 .86	T MODEL RHO-MITTC KO .05714 .00044 .00021 .05703 .05703 .05266 .04906 .04906 .04906 .04240 .03972 .03642 .03926 .03726 .03726 .03726 .03726 .03726 .03726 .03727 .04687 .04280 .03726 .	DISPLACE OOFL = 1. FRICTION JQ .713 .671 .674 .714 .755 .769 .704 .803 .836 .873 .876 .936 .936 .937 .936 .976 EPS DISPLACE OOFL = 1. FRICTION JQ	9002 USED 1-W0 -719 -803 -804 -803 -804 -805 -807 -807 -807 -807 -807 -807 -802 -806 -807 -807 -807 -802 -806 -807 -806 -807 -807 -807 -806 -807 -806 -807 -807 -807 -806 -807 -807 -807 -807 -807 -808 -808 -808	1-T .946 .863 .918 .982 .987 .926 .905 .888 .876 .897 .924 .908 .908 .940 .897 .898	RHO- EP .617 .564 .564 .566 .600 .614 .627 .666 .601 .677 .666 .691 .701 LINERHO- EP .575 .556 .591 .591	SMIP = 1. EM 1.191 1.030 1.061 1.083 1.096 1.128 1.071 1.081 1.081 1.091 1.091 1.091 1.091 1.091 1.091 1.091 1.091 1.091 1.091 1.091 1.091 1.091 1.091 1.091	.9602 ERR .896 .902 .906 .912 .906 .979 .975 .971 .985 .964 .979 .982 .961 .959 .9682 ERR .924 1.007 .991 .997
NU-MODEL PR VM (FPS) 4.05 7.27 8.52 9.95 12.19 13.43 14.91 15.91 17.29 18.48 19.92 21.25 22.63 27.31 27.95 29.80 MODEL LI NU-MODEL MODEL PR VM (FPS) 4.08 7.31 9.85 12.34 14.94 17.04 17.04 17.04	(F-5) ROPFLLER	6.600 FT = .98600 PT AME TER FMP/SMP .559 .576 .655 .677 .661 .650 .666 .652 .710 .678 .710 .710 .710 .710 .710 .710 .710 .710	= 6-000 SHP .066 .786 1.218 1.624 1.676 2.622 2.575 3.377 3.186 4.256 6.204 6.204 7.430 CONFIGURE SHD .776 .504 1.787 .504 1.787 .797 .797 .797 .797	MOI NU IN COI 1043.0 1203.0 12	DEL WETTE CF- SHIP (F- SRELATION JA 9911 8364 8089 8020 8754 9131 9337 9387 10887	D SURFACE 5) = .9 ALLOWAN KT .2520 .2943 .2943 .2945 .2918 .2661 .2541 .2541 .2541 .2517 .2014 .1793 .1714 .1553 .1516 .1553 .1516 .1553 .1516 .1553 .1516 .1553 .1516 .1553 .1516 .1553 .1516 .1553 .1516 .1529	E = 18.8600 JT .767 .760 .706 .788 .789 .805 .818 .842 .860 .892 .912 .938 .941 .941 .941 .971 .716 .687 .707 .744 .797 .744 .797 .744	920 SQ F 0000 1-WT .794 .837 .865 .873 .866 .863 .861 .866 .876 .876 .871 .882 .882 .882 .882 .883 .883 .881 .882 .883 .883 .884 .886 .896 .996 .8	T MODEL RHO-M ITTC K0	DISPLACE ODEL = 1. FRICTION JQ .713 .671 .674 .714 .755 .769 .794 .803 .836 .851 .873 .896 .936 .976 .936 .976 .936 .976 .936 .977 .678 .677 .679 .900 .976	9002 USED 1-W0 -719 -803 -804 -809 -804 -805 -807 -807 -807 -807 -806 -806 -806 -806 -806 -806 -806 -806	1-T .946 .863 .918 .882 .947 .926 .997 .926 .897 .924 .908 .940 .897 .898 341. LRS	RHO- EP .617 .564 .566 .568 .669 .677 .686 .691 .701 LINERHO- EP .575 .556 .591 .657 .665 .669 .591	SMIP = 1. EM 1.191 1.030 1.061 1.083 1.096 1.128 1.074 1.051 1.026 1.012 1.018 1.026 1.018 1.018 1.018	.9602 ERR .896 .902 .906 .912 .906 .979 .975 .907 .907 .907 .907 .908 .908 .908 .908 .908 .908 .908 .908
NU-MODEL PR VM (FPS) 4.05 7.27 8.52 9.95 12.19 13.43 14.91 15.91 17.29 18.48 19.92 21.25 22.63 27.31 27.35 29.80 MODEL PR VM (FPS) 4.08 7.31 9.85 12.34 17.04 17.04 17.04 17.04 17.04 17.04 17.04 17.04 17.04 17.04 17.04	(F-5) ROPFLLER	6.600 FT = .98600 PT AME TER FWP/SHIP ASSO .576 .559 .576 .650 .666 .652 .771 .665 .677 .777 .665 .779 .787 .685 .779 .685 .779 .685 .779 .787 .685 .779 .787 .685 .779 .787 .787 .787 .787 .787 .787 .787	= 6.000 SHP .064 .78F 1.21F 1.62F 1.67F 2.622 2.	MOI NU IN COI 1043.0 1203.0 12	DEL WETTE -SHIP (F* -SRELATION	D SURFAC 5) = .9 ALLOWAN KT .2520 .2943 .2945 .2918 .2661 .2541 .2541 .2541 .2541 .2541 .2171 .2014 .1793 .1714 .1793 .1593 .1593 .1594 .1594 .1593 .1594 .1593 .1594	E = 18.8600 JT .767 .760 .706 .788 .783 .805 .818 .842 .860 .892 .912 .938 .941 .941 .941 .941 .971 .716 .687 .707 .744 .707 .708 .708 .708 .708 .708 .708 .708	920 SQ F . 794 . 837 . 865 . 879 . 866 . 876 . 876 . 876 . 878 . 861 . 886 . 876 . 876 . 876 . 876 . 876 . 876 . 876 . 876 . 876 . 876 . 876 . 876 . 876 . 876 . 887 . 888 . 883 . 881 . 883 . 881 . 877 . 888	T MODEL RHO-MITTC K0 .05714 .06044 .06071 .05703 .05373 .05266 .05065 .04996 .04725 .04996 .03774 .03922 .03642 .03764 .03974 .04075 .04076	DISPLACE ODEL = 1. FRICTION JO .713 .671 .674 .714 .755 .769 .794 .803 .836 .851 .873 .896 .936 .936 .976 .900EL = 1. FRICTION JO .657 .689 .701 .702 .703 .704 .705 .706 .707 .707 .707 .707 .707 .707 .707	9002 USED 1-W0 -719 -803 -804 -809 -804 -805 -805 -805 -807 -807 -807 -806 -806 -806 -806 -806 -806 -806 -806	1-T .946 .863 .918 .882 .967 .926 .988 .897 .924 .908 .940 .897 .898 1-T .688 .757 .921 .891 .854 .862 .857 .888 .867	RHO- EP 617 - 556 - 556 - 556 - 556 - 601 - 627 - 686 - 657 - 686 - 677 - 686 - 681 - 701 LINERO- EP - 575 - 556 - 657 - 656 - 657 - 676 - 657 - 676 - 657 - 676	SMIP = 1. EM 1.191 1.030 1.061 1.083 1.096 1.128 1.074 1.051 1.026 1.012 1.018 1.018 1.018 EM .015 .018 .018 .018	.9602 ERR .896 .902 .906 .912 .906 .907 .907 .907 .907 .907 .907 .908 .908 .908 .908 .908 .908 .908 .908
NU-MODEL PS VM (FPS) 4.05 7.27 8.52 9.95 12.19 13.43 14.91 17.29 18.49 19.92 21.25 22.63 27.31 27.95 29.80 MODEL LI NU-MODEL PS VM (FPS) 4.08 7.31 9.85 12.34 14.94 18.23 19.89 21.30 21.30 21.30 22.81	(F-5) ROPFLLER	6.60C FT = .98600 FT = .98600 FT = .98600 FT = .559 .576 .559 .776 .661 .650 .666 .652 .710 .677 .707 .685 .710 .679 .710 .710 .710 .710 .710 .710 .710 .710	= 6.000 SHP .044 .744 .1218 1.674 1.674 2.672 2.672 2.673 3.176 4.756 6.104 6.104 6.104 6.596 7.430 COMF IGUR.	MOIN 001 NO. 1 NO. 2 NO.	DEL WETTE -SHIP (FSRELATION JA -9911 -8364 -8089 -8754 -9131 -9337 -9386 -9723 -9428 -10658 1.0658 1.1135 1.1274 -100 PFH	D SURFACE 5) = .9 ALLOWAN KT .2520 .2943 .2945 .2943 .2945 .2918 .2541 .2521 .2432	E = 1A. 8600 JT . 787 . 700 . 700 . 700 . 706 . 758 . 865 . 865 . 862 . 912 . 938 . 955 . 989 . 981 . 994 . 981 . 994 . 981 . 994 . 981 . 994 . 981 . 994 . 981 . 994 . 981 . 994 . 981 . 994 . 981 . 981 . 994 . 981 .	920 SQ F 0000 1-WT .794 .837 .865 .879 .866 .876 .863 .861 .866 .876 .8	T MODEL RHO-MITTC KO .05714 .00044 .00021 .05703 .05703 .05266 .04906 .04906 .04926 .03726 .03922 .03642 .03726 .03727 .03642 .03726 .03726 .03726 .03726 .03727 .03642 .03726 .03726 .03726 .03727 .03642 .03726 .03727 .03642 .03727 .03727 .03727 .05644 .05161 .04830 .04262 .04193 .03915 .037816	DISPLACE OOEL = 1. FRICTION JO .713 .671 .674 .714 .755 .769 .794 .803 .836 .671 .936 .930 .936 .976 .970 .906 .976 .976 .976 .977 .976 .978 .978 .978 .978 .978 .978 .978 .978	9002 USED 1-W0 -719 -803 -804 -803 -804 -805 -805 -805 -805 -805 -805 -805 -805	1-T .946 .863 .918 .982 .949 .967 .926 .905 .808 .976 .907 .924 .908 .907 .808 .1-T .608 .757 .921 .852 .857 .862 .857	RHO- EP -617 -564 -568 -600 -617 -636 -618 -617 -666 -617 -666 -701 -701 -701 -701 -701 -701 -701 -701 -701	SMIP = 1. EH 1.191 1.039 1.061 1.093 1.095 1.076 1.126 1.078 1.071 1.051 1.026 1.012 1.012 1.012 1.018 1.018 AR RATIO SMIP = 1. EH .015 .016 .017 .018 .018 .018 .018 .018 .018 .018 .018	.9602 ERR .896 .962 .962 .966 1.012 .979 .982 .975 .961 .979 .982 .961 .950 .960 .960 .960 .960 .960 .960 .960 .96
NU-MODEL PR VM (FPS) 4.05 7.27 8.52 9.95 12.19 13.43 14.91 15.91 17.29 18.48 19.92 21.25 22.63 27.31 27.35 29.80 MODEL PR VM (FPS) 4.08 7.31 9.85 12.34 17.04 17.04 17.04 17.04 17.04 17.04 17.04 17.04 17.04 17.04 17.04	(F-5) ROPFLLER	6.60C FT = .98600 FT = .98600 FT = .98600 FT = .559 .576 .559 .776 .661 .650 .666 .652 .710 .677 .707 .685 .710 .679 .710 .710 .710 .710 .710 .710 .710 .710	= 6.000 SHP .064 .78F 1.21F 1.62F 1.67F 2.622 2.	MOI NU IN COI 1043.0 1203.0 12	DEL WETTE -SHIP (F* -SRELATION	D SURFAC 5) = .9 ALLOWAN KT .2520 .2943 .2945 .2918 .2661 .2541 .2541 .2541 .2541 .2541 .2171 .2014 .1793 .1714 .1793 .1593 .1593 .1594 .1594 .1593 .1594 .1593 .1594	E = 18.8600 JT .767 .760 .706 .788 .783 .805 .818 .842 .860 .892 .912 .938 .941 .941 .941 .941 .971 .716 .687 .707 .744 .707 .708 .708 .708 .708 .708 .708 .708	920 SQ F . 794 . 837 . 865 . 879 . 866 . 876 . 876 . 876 . 878 . 861 . 886 . 876 . 876 . 876 . 876 . 876 . 876 . 876 . 876 . 876 . 876 . 876 . 876 . 876 . 876 . 887 . 888 . 883 . 881 . 883 . 881 . 877 . 888	T MODEL RHO-MITTC K0 .05714 .06044 .06071 .05703 .05373 .05266 .05065 .04996 .04725 .04996 .04725 .04996 .03774 .03922 .03642 .03764 .03974 .04076	DISPLACE ODEL = 1. FRICTION JO .713 .671 .674 .714 .755 .769 .794 .803 .836 .851 .873 .896 .936 .936 .976 .900EL = 1. FRICTION JO .657 .689 .701 .702 .703 .704 .705 .706 .707 .707 .707 .707 .707 .707 .707	9002 USED 1-W0 -719 -803 -804 -809 -804 -805 -805 -805 -807 -807 -807 -806 -806 -806 -806 -806 -806 -806 -806	1-T .946 .863 .918 .882 .967 .926 .988 .897 .924 .908 .940 .897 .898 1-T .688 .757 .921 .891 .854 .862 .857 .888 .867	RHO- EP 617 - 556 - 556 - 556 - 556 - 601 - 627 - 686 - 657 - 686 - 677 - 686 - 681 - 701 LINERO- EP - 575 - 556 - 657 - 656 - 657 - 676 - 657 - 676 - 657 - 676	SMIP = 1. EM 1.191 1.030 1.061 1.083 1.096 1.128 1.074 1.051 1.026 1.012 1.018 1.018 1.018 EM .015 .018 .018 .018	.9602 ERR .896 .962 .966 .912 .976 .977 .982 .985 .964 .979 .982 .964 .959 .954 .9682 ERR .924 1.007 .969 1.053 .976

Appendix 3 - Propulsive Characteristics for 100 Per Cent Tunnel Hull

CONFIGURATION 3 - 100 PER CENT TUNNELS - 5.25 INCH PROPELLERS

MODEL LENGTH . 6.600 FT MU-MODEL (E-5) . 1.02600 MODEL PROPELLER DIAMETER .			• 5.250	NU-	EL WETTER SHIP (F.	5) - 1.0	2600		RHO-M	DISPLACE ODEL = 1 FRICTION	.9620	341. LB		EINEAR RATIO = 1.00				
HUUEL	-worefren	DIATIER	. 3.536	14 604	ME LA LIGH	ALLOWA	CE -0.00	000	1110	- HICITON	OSED							
VM (FPS)	EHD	EMP/SHP	SHP	RPH	JA	KT	JT	1-41	KO	٥ر	1-00	1-1	EP	EH	ERR			
7.29	.276	.356	.775	1377.2	.7259	.4133	.718	. 989	.08921	.725	.998	.864	.525	.672	1.008			
8.50	.496	.391	1.270	1620.3	.7194	.405A	.733	1.019	.08972	.719	1.000	.755	.535	.741	.986			
10.02	.776	.475	1.635	1796.8	.7648	.3877	.769	1.006	.08475	.772	1.009	.052	.558	.847	1.003			
12.44	1.165	.516	2.258	2014.1	.8476	. 3554	.835	. 986	.08307	.789	.932	.894	.599	.908	.949			
14.54	1.473	.520	2.788	2191.6	.9094	.3431	.860	.945	.07960	.426	.908	.846	.614	.895	.960			
15.01	1.545	.516	2.995	2258.A	.9113	.3417	.863	.947	.07813	.442	.924	.013	.616	.859	.975			
16.99	1.636	.522	3.522	2423.9	.9613	. 3148	.919	.955	.07435	.883	.918	.805	.648	.843	.956			
19.92	2.307	.572	4.037	2615.3	1.0446	.2772	.997	.955	.06784	.954	.913	.841	.689	.861	.941			
22.95	2.096	.633	4.578	2824.3	1.1144	.2469	1.061	.952	.06109	1.027	.922	.082	.720	.927	.949			
25.00	3,395	.430	5.324	2994.8	1.1448	.2339	1.089	.951	.05963	1.043	.911	.881	.732	.926	.929			
27.42	3.963	.640	6.119	3177.1	1.1836	.2202	1.118	.945	.05736	1.068	.902	.896	.743	.948	.919			
29.97	4.713	.647	7.283	3419.5	1.2020	.2100	1.140	.948	.05476	1.096	.912	.882	.751	.930	.926			
35.04	4.517	.629	10.356	3899.9	1.2322	.1972	1.167	.947	.05249	1.121	.909	.854	.761	.902	.918			

CONFIGURATION 7 - 100 PEN CENT TUNNELS - 5.25 INCH PROPELLERS

NU-HODE	ENGTH = 'L (F+5) = ROPELLER		5.250	NU-	SHIP IF.	5) = .91	9400		RHO-M	DISPLAC ODEL = 1 FRICTION	.9602	341. LRS		R RATIO	• 1.000 .9602
VM (FPS)	EHP	EMP/SHP	SHP	чрм	JA	KT	JI	1-41	KQ	٥ر	1-40	1-1	EP	EH	ERR
3.84	.028	.542	.056	604.2	.8716	.ZR52	.940	1.125	.075A5	.467	.994	.963	.681	.856	.961
14.89	1.527	.440	7.468	2704.9	.7539	. 1249	1-108	1.470	.05250	1.120	1.486	.857	.739	.5A3	1.022
17.23	1.873	.538	3.485	2424.2	.4727	. 30 35	.442	.969	.07316	.896	.921	.837	.661	.864	.941
24.35	3.201.	.674	4.750	2914.5	1.145A	. 2245	1.109	.96A	.05773	1.064	.928	.950	.740	.982	.928

CONFIGURATION 4 - 100 PER CENT TUNNELS - 6.0 INCH PROPELLERS 10 DEG TRIM TABS

NU-MOI	DEL (E+5)	6.400 FT 1.02600 014METER	060	NU-	SHIP IF+	SUMFACE 5) = 1.02 ALLOWANC						341. LAS	LINEAR RATIO = 1.000 BHO-SMIP = 1.9620		
VMIFPS	FHP	FHP/SHP	SHP	HPM	JA	KT	JT	1-41	KO	JO	1-40	1-1	EP	EH	ERR
4.10	.012	.414	77	507.1	.8350	.3774	.612	.731	.05894	.490	.824	.543	.505	.743	1.105
6.76	-141	.530	.341	907.0	.8349	. 3076	.673	. RO2	.05816	.700	. 834	.750	.547	.935	1.037
8.18	.384	.512	.749	1247.1	.7871	.3094	.669	.850	.05955	.683	-867	.787	.544	.925	1.018
9.90	.617	.550	1.123	1444.4	.8225	.2957	.697	. 448	.05747	.709	.862	.816	.562	.962	1.016
12.51	.967	.531	1.422	17.7.1	.A742	. 2744	.741	.848	.05547	.734	.839	.771	.590	.909	.989
14.94	1.347	.570	2.363	1910.5	.9364	.2572	.777	ASA.	.05225	. 774	.825	.776	.611	.937	.996
17.18	1.768	.546	3.417	2112.2	.9701	. 6374	.818	. A 3A	.04937	10	.830	.785	.633	.937	.986
19.39	2.239	.614	1.636	2748.3	1.010A	5+15.	.855	.941	.04678	. 442	.828	.012	.652	.966	.978
21.22	2.674	.656	4.674	2437.4	1.0447	. 2044	.885	.847	.04338	.AHS	.A47	.836	.666	.987	.999
23.03	3.149	.651	4.440	2507.6	1.0600	.1940	.909	.850	.04308	. ABA	.832	.850	.675	1.000	.964
24.81	1.685	.651	5.656	2746.7	1.0939	.1480	.920	.449	.04210	.900	.831	.845	.680	.996	.962
27.25	4,558	.646	6.040	2965.3	1.1007	.1797	.938	.850	.03927	.935	.848	.855	.686	1.005	.995

CONFIGURATION 5 - 100 PER CENT TUNNELS - 6.0 INCH PROPELLERS 5 DEG THIM TARS

	LENGTH =	6.400 FT			EL WETTE			20 50 6		DISPLACE		341. LBS		AR PATIO	
	PROPFLLER		. 6.000				NCF =0.00000			ITTC FRICTION USED			N 40-	2416 a 1	. 4620
VM (FPS)	EHP	EHP/SHP	SHP	-	JA	*1	JT	1-#1	*0	,10	1-40	1-7	FP	FH	FRR
4.10	.032	.512	.040	546.9	.8947	.2678	.755	.839	.05685	.716	.796	.788	.598	.939	.946
6.32	.407	.500	.815	1274.9	.7819	. 30 35	.682	.A72	.05034	.673	.860	.798	.552	.915	.988
9.82	.616	.493	1.256	1440.2	.7904	.2964	.696	.AHO	.05A23	.490	.884	.769	.562	.874	1.004
9.96	.630	.552	1.142	1451.4	. A223	.2935	.702	.854	.05735	.710	.864	.024	.565	.965	1.011
11.88	.920	.612	1.503	1624.6	. 4757	.26m3	.754	.861	.05365	.757	.864	.879	.597	1.020	1.004
15.00	1.416	.597	2.376	1934.0	.9747	.2477	.794	.855	.05065	.794	.853	.824	.622	.963	.997
17.10	1.778	.625	2.845	2117.1	9739	.2244	. 434	.856	.04690	.841	.863	.824	.642	.962	1.012
19.53	2.223	.610	1.642	2318.3	1.0100	5015	. 474	.865	.04507	. 864	.854	.013	.661	.961	.981
21.22	2,545	.634	4.079	2446.6	1.0448	.1991	. 897	.862	.04295	. 490	.855	.825	.671	.957	.987
22.80	2.944	AAA.	4 3"	2559.1	1.0441	.1469	.927	.867	.04083	.915	.857	.867	.682	1.000	.979
24.61	3,454	.658	5.250	27.0.2	1.0007	.1779	.941	.867	.04022	.924	.851	.961	.6A7	.993	.964
27.26	4.247	.671	A. 19F	29-1-6	1.10-1	.1736	.957	.863	.03A31	.947	.855	.855	.492	.990	.980
30.01	5,358	. 641	7.470	31 46 . 7	1.1265	.1501	975	.865	.03715	500	.854	.070	.697	1.006	.971
34,48	7,523	. 477	11.115	3610.1	1.1991	.1544	.991	.854	.03643	.970	.847	.875	.700	1.013	.954

Appendix 3 - (Cont.)

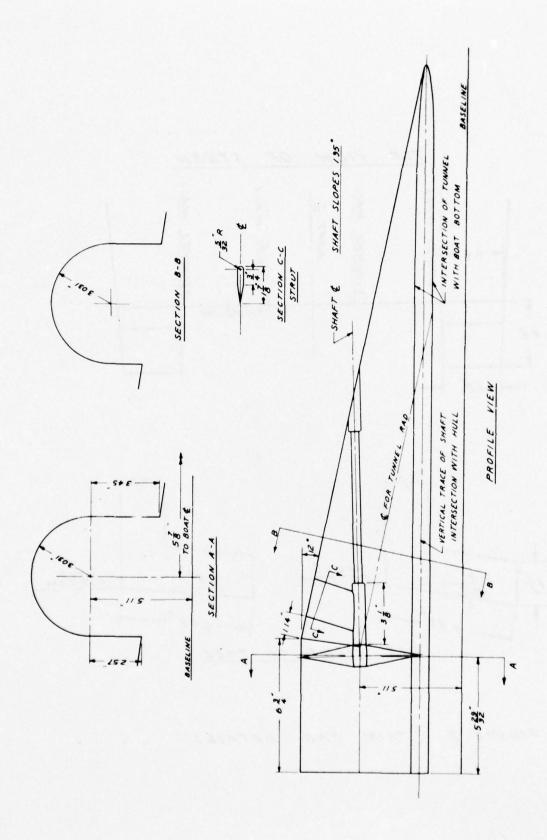
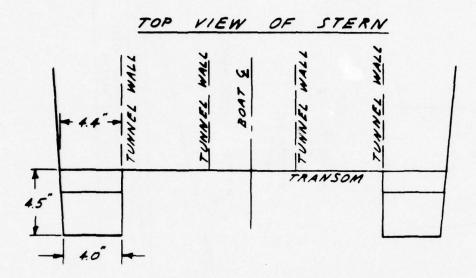


Figure 1 - Details of 100 Per Cent Tunnel



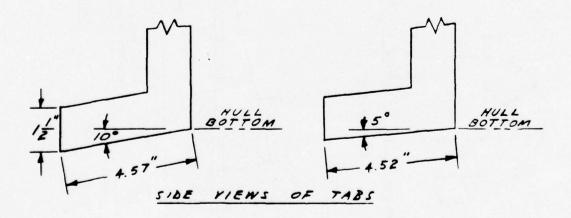
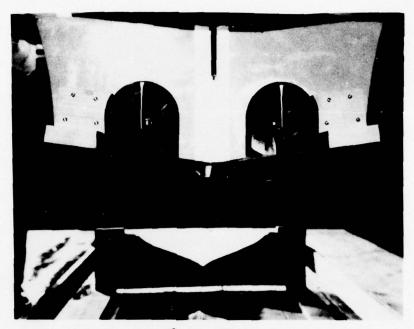
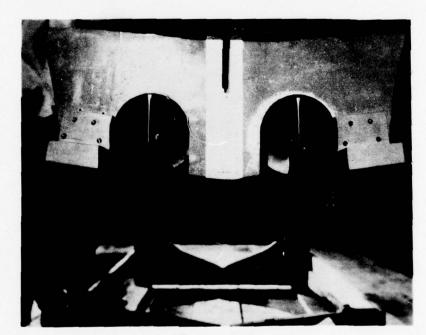


FIGURE 2. TRIM TAB DETAILS



5° Trim Tabs



10° Trim Tabs

Figure 3 - Stern Views Showing Trim Tabs, 6 Inch Propellers and Appendages

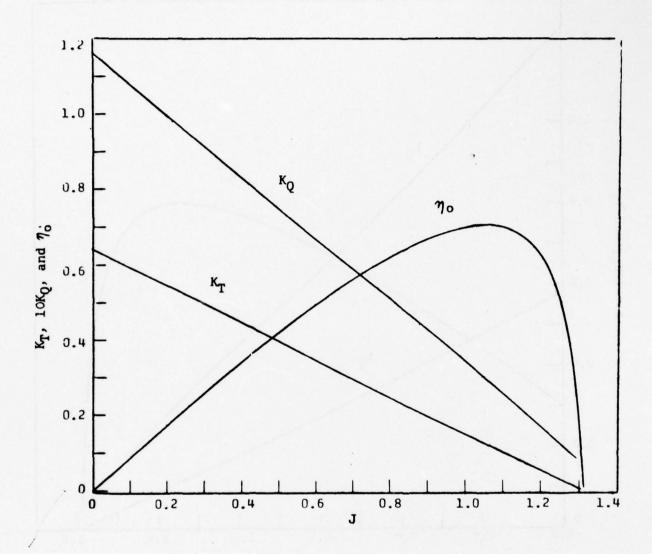


Figure 4(a) - Open Water Propeller Characteristics for the 6.0 Inch Diameter Propeller

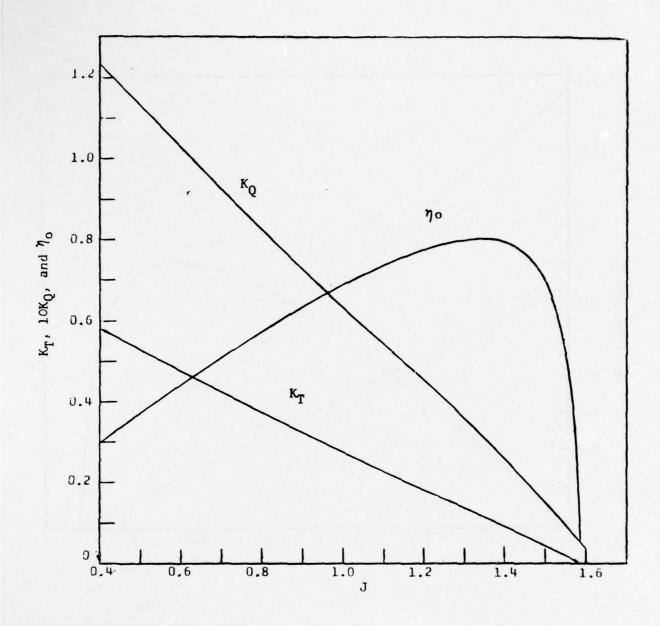


Figure 4(b) - Open Water Propeller Characteristics for the 5.25 Inch Diameter Propeller

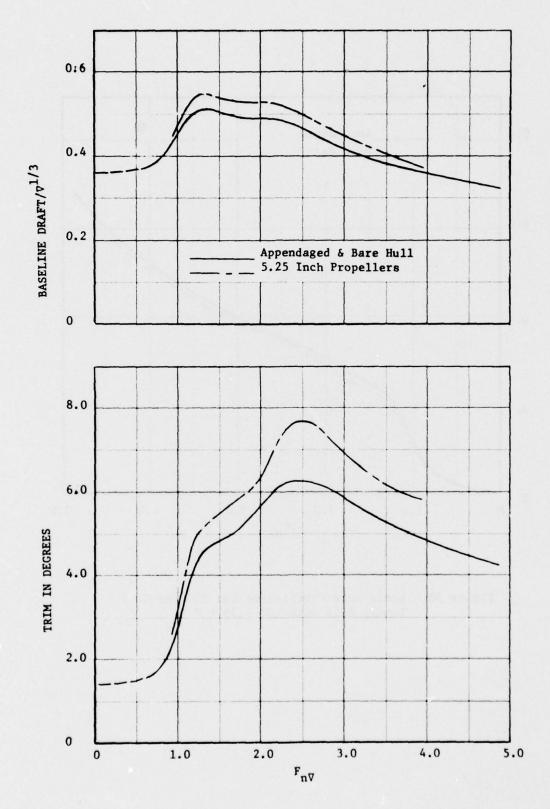


Figure 5A - Trim Angle and Baseline Draft of 100 Per Cent Tunnel Hull with LCG = 39.8% $L_{\rm p}$

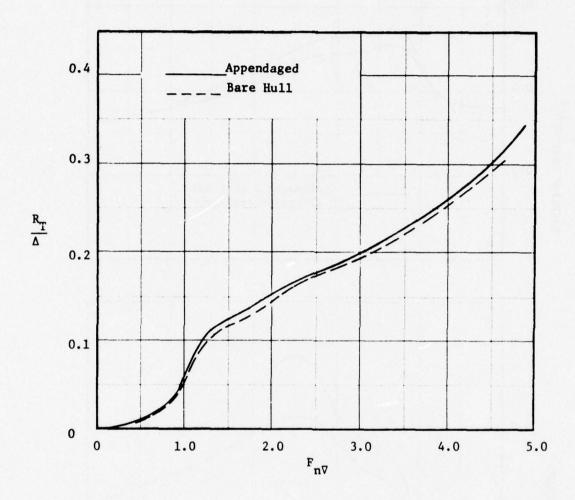


Figure 5B - Resistance Coefficient for 100 Per Cent Tunnel Hull with LCG = 39.8 % L

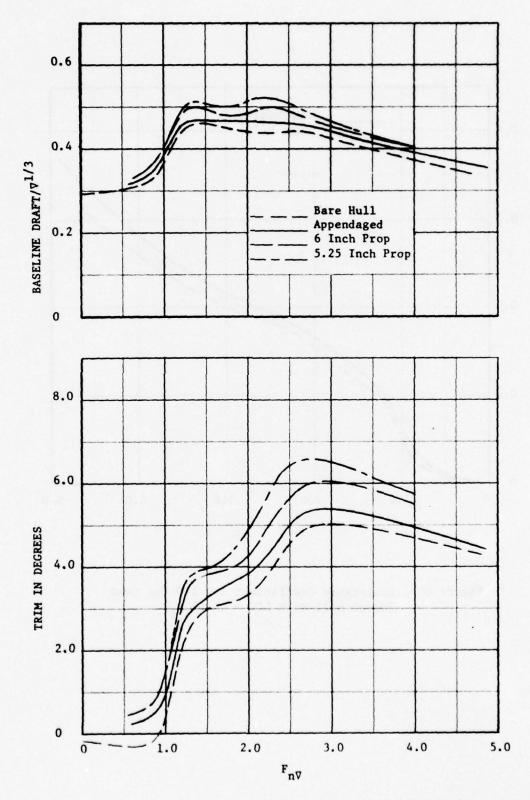


Figure 6A - Trim Angle and Baseline Draft of 100 Per Cent Tunnel Hull with LCG = 44.8% $L_{\rm p}$

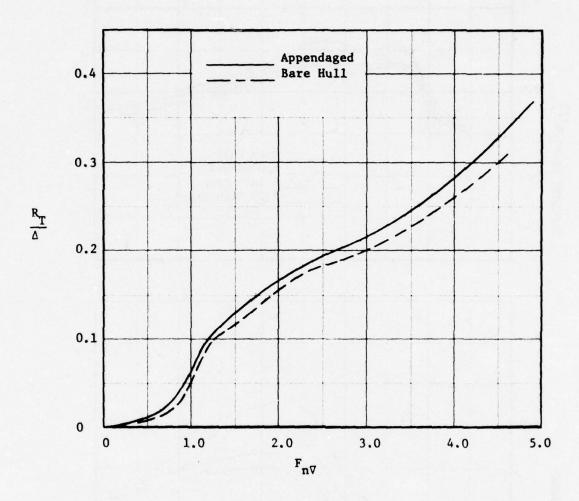


Figure 6B - Resistance Coefficient for 100 Per Cent Tunnel Hull with LCG = 44.8% L_p

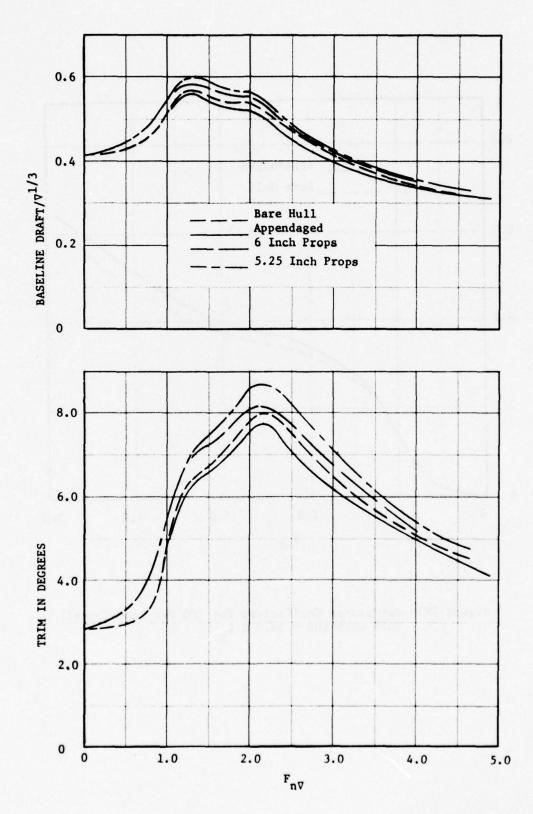


Figure 7A - Trim Angle and Baseline Draft of 100 Per Cent Tunnel Hull with LCG = 34.8% L $$\rm p$$

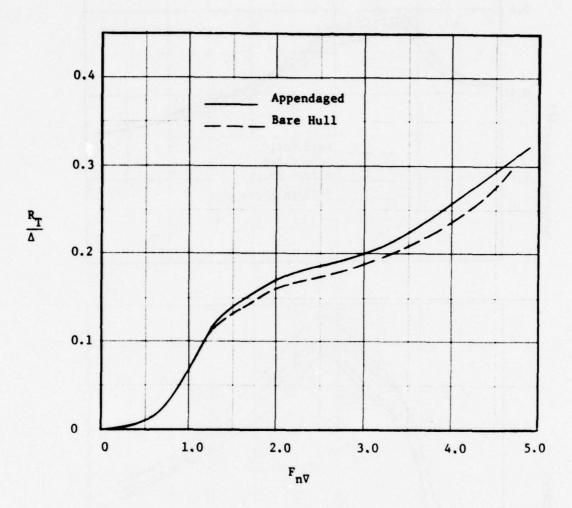


Figure 7B - Resistance Coefficient for 100 Per Cent Tunnel Hull with LCG = 34.8 % L $_{\rm p}$

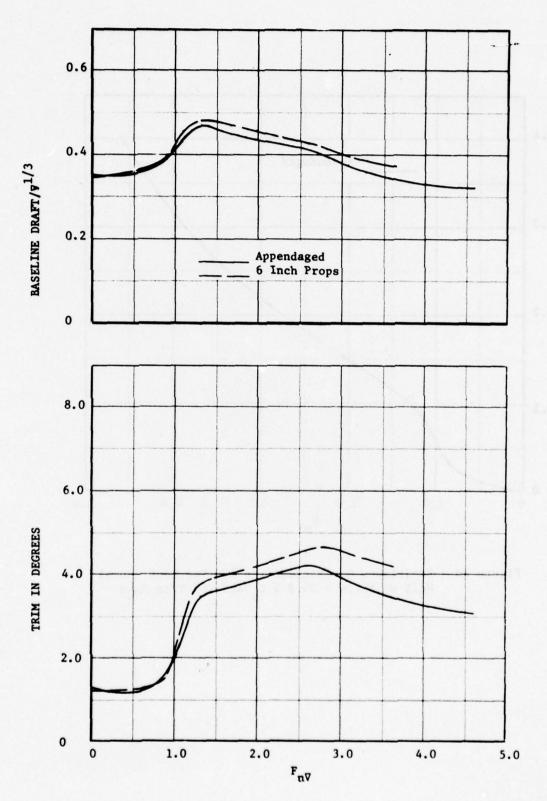


Figure 8A - Trim Angle and Baseline Draft of 100 Per Cent Tunnel Hull with LCG = 39.8 % $\rm L_p$ and 10^{\odot} Trim Tabs

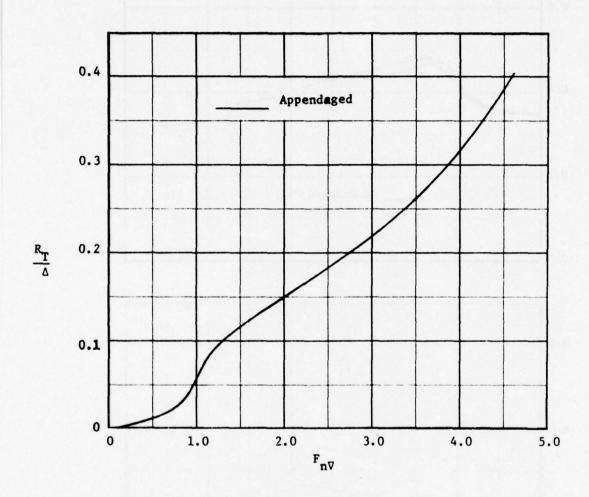


Figure 8B - Resistance Coefficient for 100 Per Cent Tunnel Hull with LCG = 39.8 % L and 10° Trim Tabs

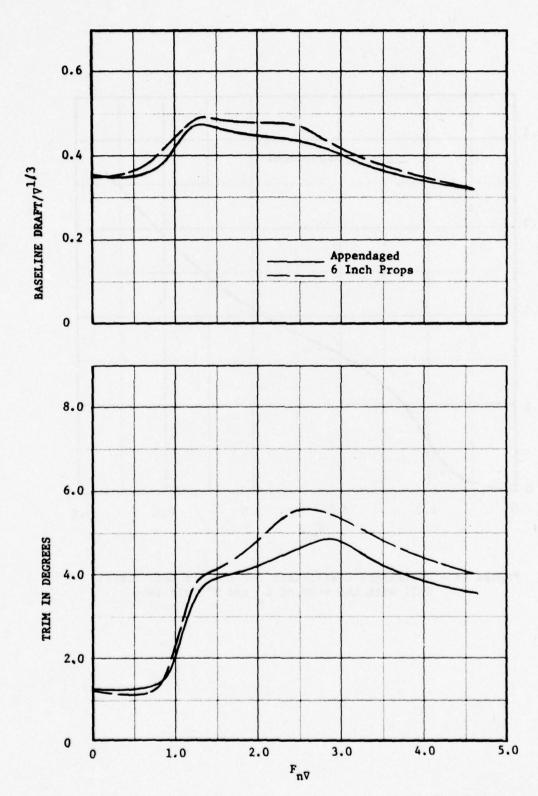


Figure 9A - Trim Angle and Baseline Draft of 100 Per Cent Tunnel Hull with LCG = 39.8% L p and 5° Trim Tabs

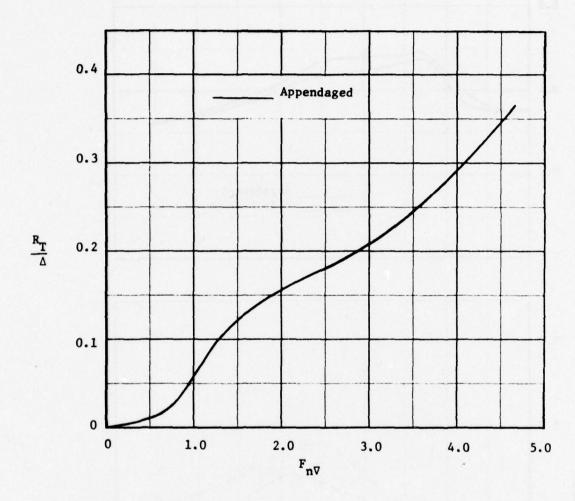


Figure 9B - Resistance Coefficient for 100 Per Cent Tunnel Hull with LCG = 39.8% L and 5° Trim Tabs

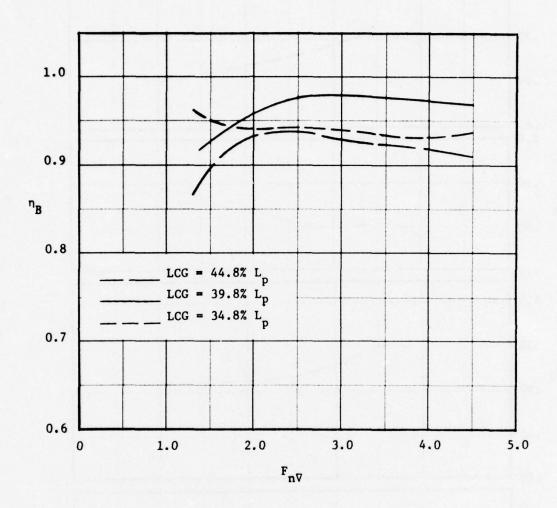


Figure 10 - Appendage Drag Factor $\boldsymbol{\eta}_{B}$ for 100 Per Cent Tunnel Hull

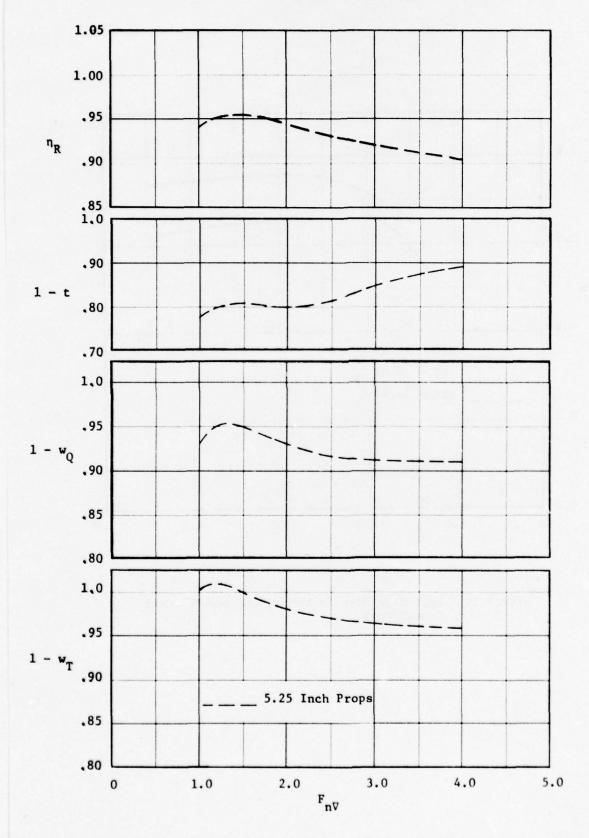


Figure 11 - Propulsive Characteristics: LCG = 39.8% L_p

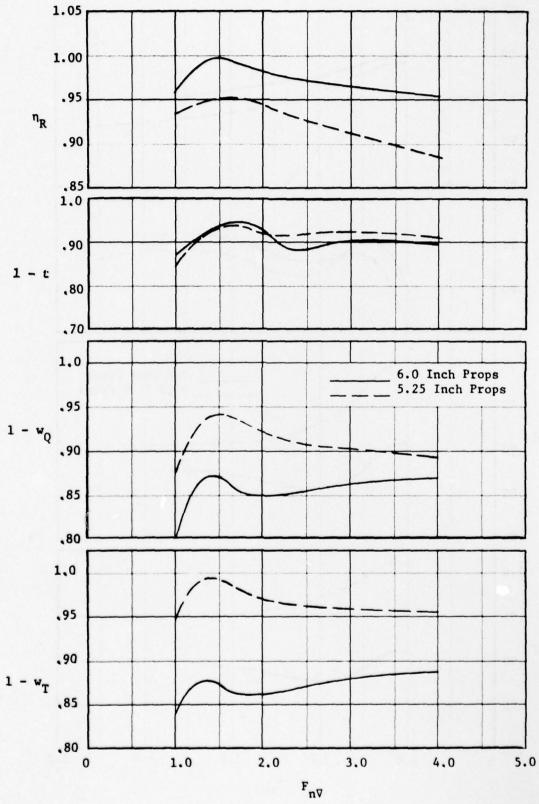


Figure 12 - Propulsive Characteristics: LCG = 44.8% L

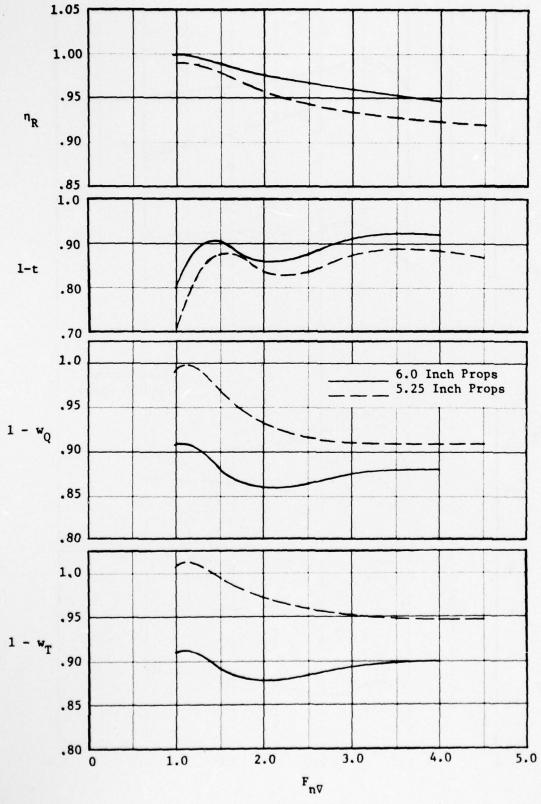


Figure 13 - Propulsive Characteristics: LCG = 34.8% Lp

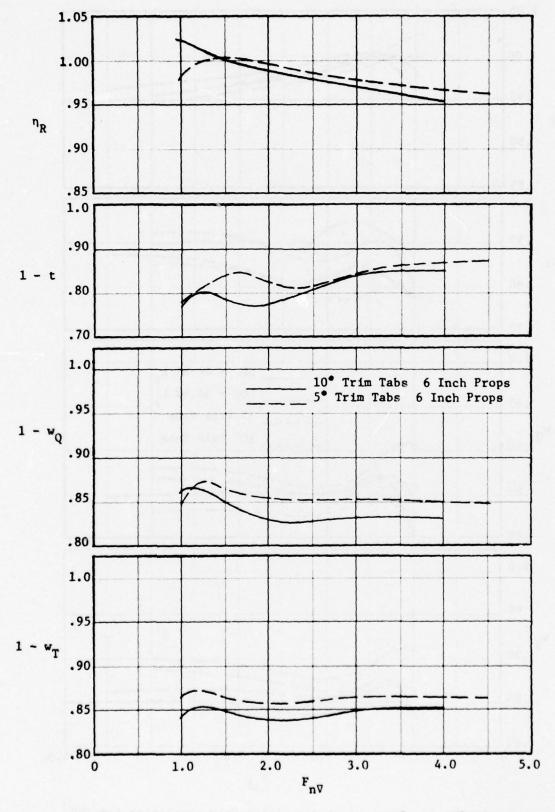


Figure 14 - Propulsive Characteristics: 5° and 10° Trim Tabs

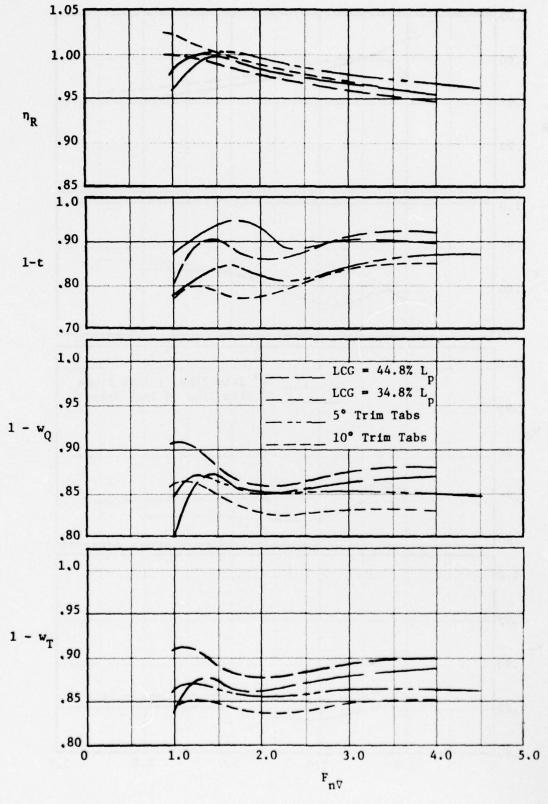


Figure 15 - Summary of Propulsive Characteristics for 100 Per Cent Tunnel Hull with 6 Inch Propellers

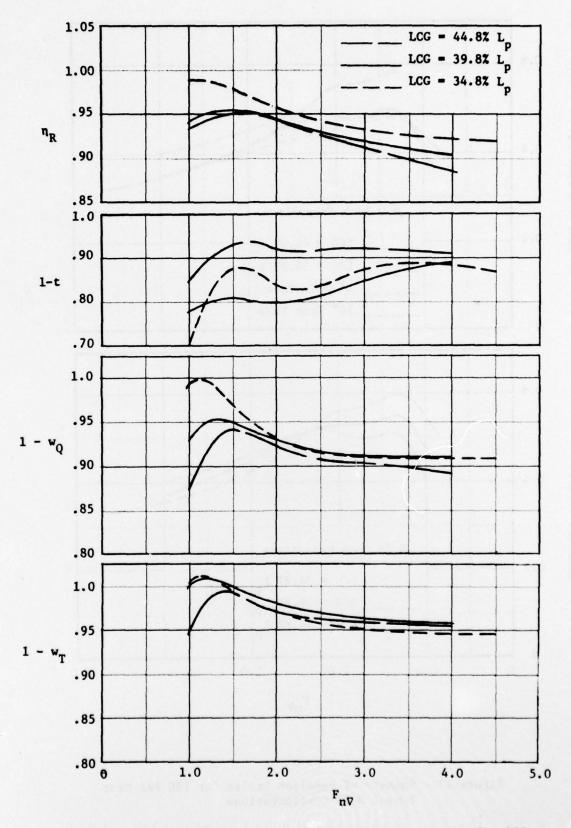


Figure 16 - Summary of Propelsive Characteristics for 100 Per Cent Tunnel Hull with 5.25 Inch Propellers

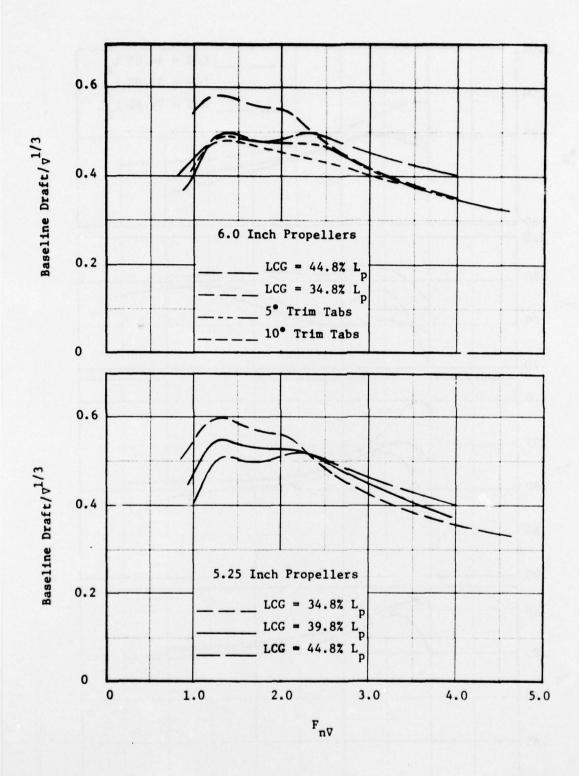


Figure 17 - Summary of Baseline Drafts for 100 Per Cent Tunnel Hull Configurations

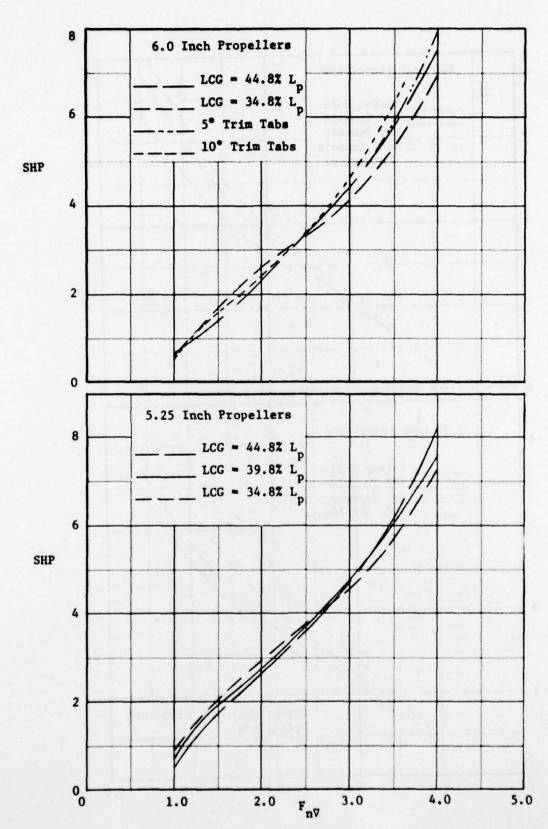


Figure 18 - Summary of Shaft Power Requirements for 100 Per Cent Tunnel Hull Configurations

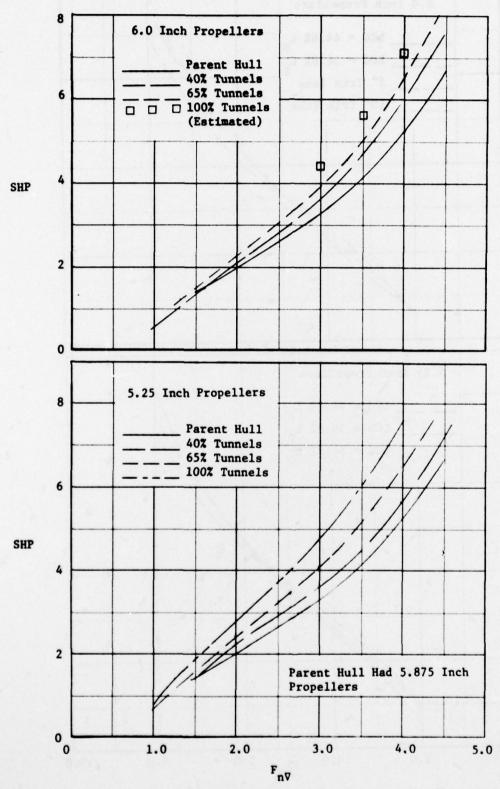


Figure 19 - Shaft Power Required For Parent and Tunnel Hull Craft

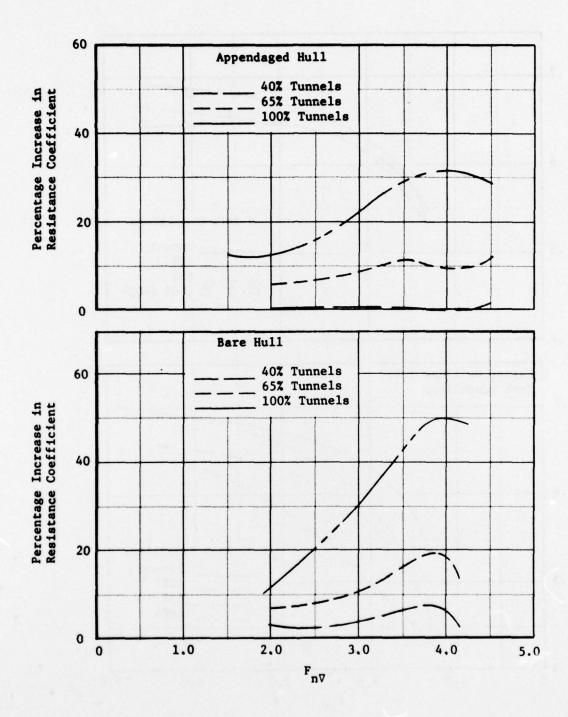


Figure 20 - Percentage Increase in Resistance Coefficient Over Parent Hull for Three Tunnel Depths

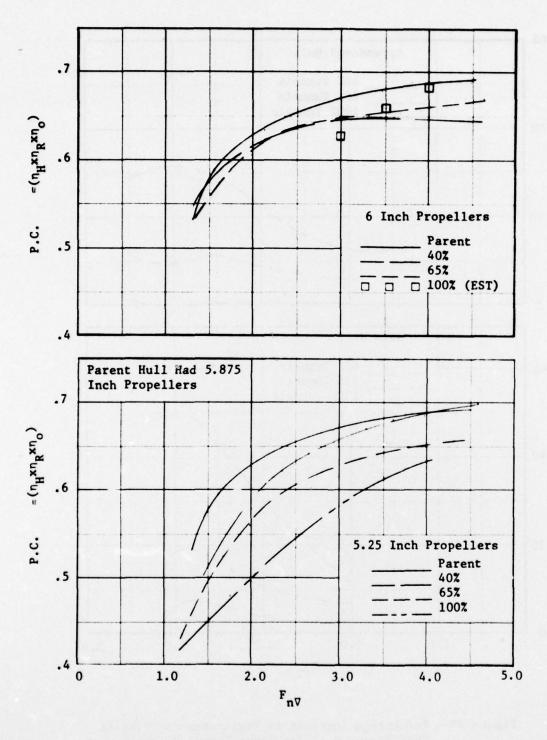


Figure 21 - Summary of Propulsive Coefficients for Parent and Tunnel Hulls

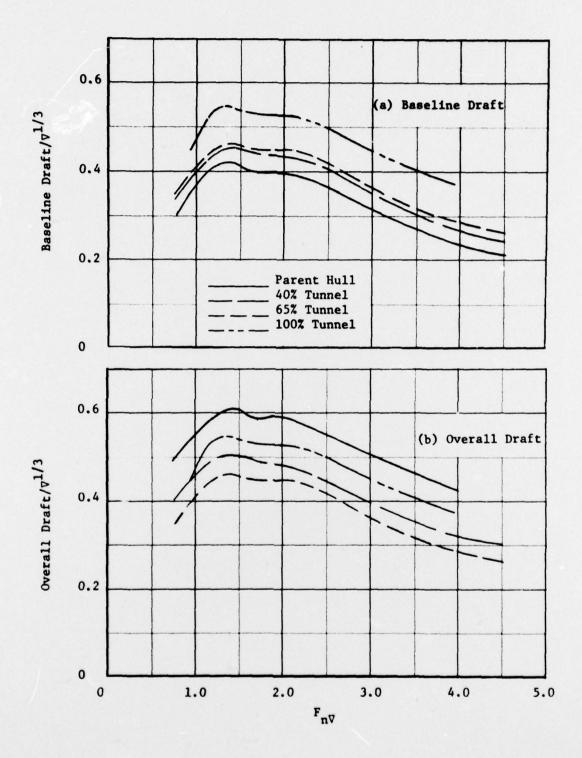


Figure 22 - Baseline Draft and Overall Draft for Parent and Tunnel Hull Craft

ADDENDUM

CORRECTIONS AND ADDITIONS TO "AN EXPERIMENTAL STUDY OF A HIGHPERFORMANCE TUNNEL HULL CRAFT" BY K. HARBAUGH AND D.L. BLOUNT,
SNAME, APRIL 1973.

Since the above paper was published, errors have been discovered in Figures 10 and 16 of that report. At the request of the authors, corrected versions of these figures are presented here. Additional data for the 100% tunnel hull are included for completeness.

The original Figure 10 was intended to illustrate the variation in η_A with F_n . This factor allows for the conversion of bare parent hull resistance data to resistance of the appendaged tunnel hull. In fact, the curves given in the original Figure 10 represent η_B , the ratio of bare to appendaged hull resistance for the same hull form. The relationship can be expressed as

$$\eta_{A} = \frac{R_{T_{BP}}}{R_{T_{AX}}}$$

$$= \frac{R_{T_{BP}}}{R_{T_{BX}}} \times \frac{R_{T_{BX}}}{R_{T_{AX}}}$$

$$= \eta_{C} \times \eta_{B}$$

Obviously, then the curves are in error by a factor η_C , whose magnitude can be significant. Figure 7 of Reference 1 presents curves of $(\frac{1}{\eta_C}-1)$ x 100 and $(\frac{1}{\eta_D}-1)$ x 100. These have been reworked and supplemented with 100% tunnel results and are given in Figure Al, as curves of η_C and η_D vs F. This figure illustrates the magnitude of the error in the original curves of η_A . The error was carried through into the determination of the efficiency factor

 $(\frac{1}{n_A x n_H x n_R})$ depicted as Figure 16 of Reference 1 and accounts for the more optimistic conclusions of Reference 1 compared to the current report as to the merits of the various tunnel hull forms.

Corrected Figures 10 and 16 for Reference 1 are appended as Figures A2 and A3, with additional data for the 100% tunnel hull. These figures can be used with the preliminary design method outlined in Reference 1 to obtain speed-power estimates based on parent hull speed-bare hull resistance relationships.

Notation

 $R_{T_{AX}}$

Appendaged hull resistance: X=P for parent hull

X=T for tunnel hull

RTDY

Bare hull resistance: X=P for parent hull X=T for tunnel hull

$$\eta_{A} = \frac{R_{T_{BP}}}{R_{T_{AX}}}$$

$$n_{B} = \frac{R_{T_{BX}}}{R_{T_{AX}}}$$

$$\eta_C = \frac{R_{T_BP}}{R_{T_BX}}$$

$$\eta_{D} = \frac{R_{TAP}}{R_{TAY}}$$

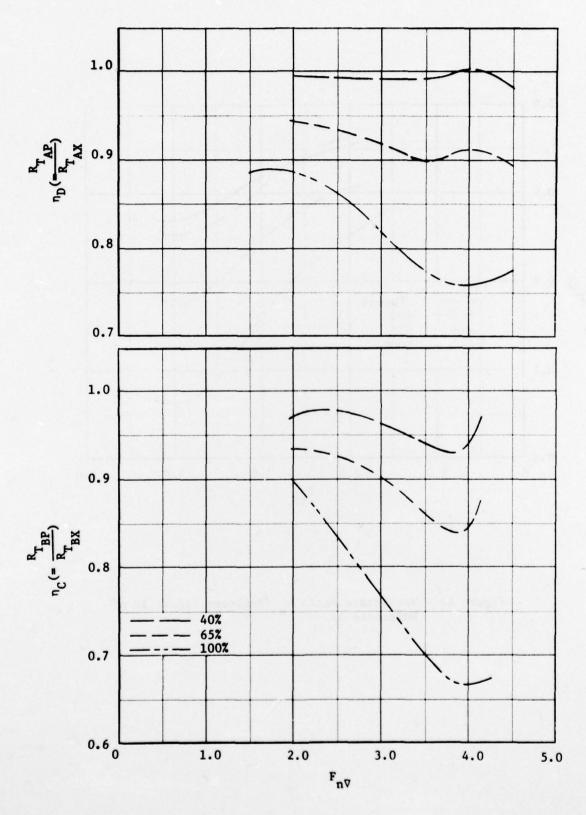


Figure Al - Resistance Ratios $\boldsymbol{\eta}_{C}$ and $\boldsymbol{\eta}_{D}$ for Three Tunnel Hulls

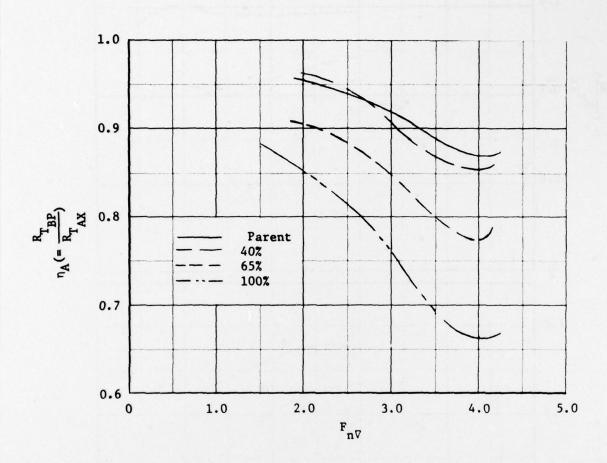


Figure A2 - Resistance Ratio $n_{\mbox{\scriptsize A}}$ (Replaces Figure 10 of Reference 1)

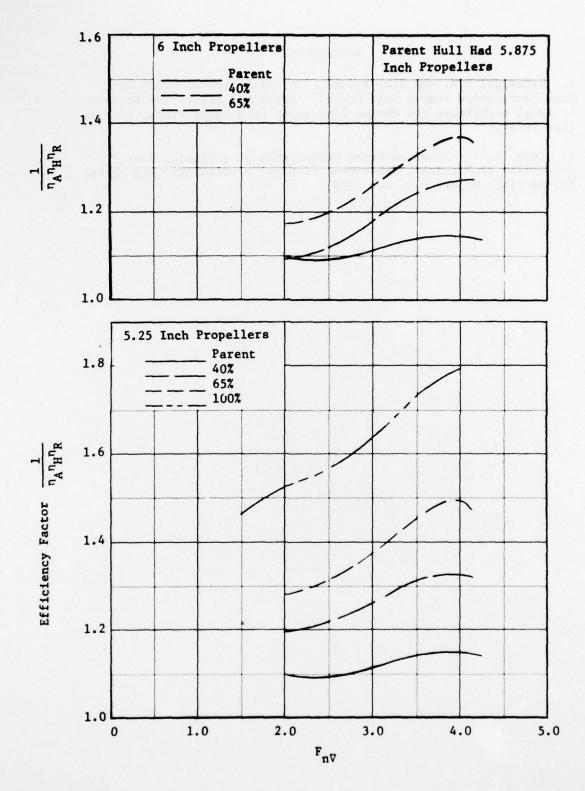


Figure A3 - Efficiency Factor $\frac{1}{{}^{\eta}A^{\eta}H^{\eta}R}$ (Replaces Figure 16 of Reference 1)

REFERENCES

- 1. Harbaugh, K.H. and D.L. Blount, "An Experimental Study of a High Performance Tunnel Hull Craft," presented before the Society of Naval Architects and Marine Engineers, Lake Buena Vista, Florida (Apr 1973).
- 2. Toro, A.I., "Shallow-Water Performance of a Planing Boat," University of Michigan, Department of Naval Architecture and Marine Engineering, Report 019, Apr 1969.

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